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**Optimizing HVAC Systems using Occupant
Detection and User Thermal Preferences**

A thesis submitted in partial satisfaction

of the requirements for the degree

Master of Science in Electrical Engineering and Computer Science

by

Alex Beltran

2017

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2017

The thesis of Alex Beltran is approved.

Shawn Newsam

Miguel Á. Carreira-Perpiñán

Alberto E. Cerpa, Committee Chair

University of California, Merced

2017

Dedicated to my mother, father, and all three of my big brothers.

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VITA

- 2008-2012 Bachelor of Science, Computer Science
- Teaching Assistant, Electrical Engineering and Computer Science, Department, University of California–Merced.
- 2013–2017 Research Assistant, Electrical Engineering and Computer Science, University of California–Merced.

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ABSTRACT OF THE THESIS

Optimizing HVAC Systems using Occupant Detection and User Thermal Preferences

by

Alex Beltran

Master of Science in Electrical Engineering and Computer Science

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Professor Alberto E. Cerpa, Chair

Buildings are a crucial part of our daily lives and people spend 87% of their time inside buildings. To maintain thermal comfort in buildings a significant amount of energy is used to condition these spaces. In the US buildings account for 40% of energy usage and of that 50% of energy goes to heating, ventilation, and air conditioning (HVAC). Often this energy is wasted by conditioning empty rooms or by leaving building occupants unsatisfied with the temperature of their room.

In this thesis we present several ways to reduce energy usage while improving user comfort. First, we reduce energy consumption by incorporating a new thermal-based occupancy sensor. Energy can be saved by using these thermal based sensors to detect occupancy and predict movements between rooms and only conditioning rooms which are occupied. Second, we focus on improving occupant's thermal comfort by giving them a method of participatory voting and influencing how they vote by using several feedback mechanisms which can increase user engagement and reduce HVAC energy usage. And finally, we combine the previous concepts into an optimization problem that finds the optimal control sequences based on occupancy, user voting, and several other inputs.

CHAPTER 1

Introduction

Internet of Things (IoT) has become ubiquitous at home and in offices both in academia and commercially. Wireless sensor mesh networks can be established to gather sensor data which is sent to a gateway node and sent to a server. Users then interact with these devices and their environment using IoT.

According to the US Department of Energy, buildings account for 40% of energy usage in the U.S. and, of that 40%, 50% of this energy is used for heating, ventilation, and air conditioning (HVAC) [bed]. Conditioning buildings is important since people spend 87% of their time inside a building [KNO01]. The goal of conditioning office spaces is largely missed since 35% of occupants report that they are dissatisfied with their thermal comfort [EC]. In addition to uncomfortable occupants, a common issue is that spaces are conditioned when they are unoccupied.

In this thesis, we contribute the following:

- A novel occupancy detection method using low power thermal array sensors in a wireless sensor network
- Develop a voting application to allow users to vote on their thermal preference
- Evaluate 5 different voting feedback systems and their influences on how

users vote and the changes in energy usage

- Develop and evaluate the combination of occupancy detection, occupancy prediction, and thermal comfort into a thermal model to calculate optimal control sequences

This thesis begins at Chapter 2 with related work for all of the topics. After Chapter 2, this thesis can be broken into three core chapters: Chapter 3 covers occupancy detection, Chapter 4 covers participatory sensing, and Chapter 5 combines the previous two technologies to create a model using occupancy detection and comfort. The final chapter is left for closing remarks and conclusions.

CHAPTER 2

Related Work

2.1 Occupancy Detection

The authors of [KJD09] test the use of cameras as optical turnstiles to estimate occupancy. However, they fail to address how cumulative error can impact occupancy estimates. As previously mentioned, even a single error will cause error to propagate forward. The total ground truth was also limited to 4 hours total for different times of the day.

The authors of [EAC] also use cameras as optical turnstiles. They mount multiple strategically placed cameras in hallways to measure occupancy for several areas. In addition, they also have a PIR sensor network in order to better distinguish empty rooms. Again, the major drawback to this approach is the cumulative error that will occur. However, unlike [KJD09], they discuss strategies for reducing cumulative error. They impose maximum occupancy limits and use a particle filter with an occupancy model with live data to estimate the error. However, since their approach uses a model, their approach also requires a non-trivial amount of ground truth occupancy data (2 weeks), collected using webcams. The occupancy RMSE achieved was 1.83 persons, more than 5 times the ThermoSense's error.

In [BLT10], elderly people are tracked using Imote2 motes with Enalab cameras and utilizing a motion histogram for a period of 1 week. Occupant counts are

achieved by counting peaks within the histograms. Their system also includes PIR sensors in order to detect occupancy for certain areas. There are several drawbacks to this approach. Since the camera must continually poll the room in order to generate the motion histograms, the power consumption during periods of occupancy will be high. The problem of privacy also exists for this system; cameras must be placed directly in the room.

The papers [LSS10, ABD11] describe methods that uses door sensors with PIR sensors to obtain a binary measurement of occupancy. Ground truth data was collected using surveys and manual annotations, and the experiments were run for 1-4 weeks and 4 days respectively. By adding door sensors, they minimize instances where overly still occupants become invisible to the PIR sensor. While this technique improves the binary measurement of occupancy, these systems do not provide a precise occupancy estimate.

Rather than rely on PIR sensors alone, the authors of [SBK11] also utilize active radio frequency identification (RFID) tags to determine occupancy. By deploying multiple antennas within the occupied space, the RFID tags were able to broadcast their presence every 5 seconds. The limitation to this strategy is that every occupant must possess a RFID tag, and that tag must always be co-located with the occupant.

The papers [LHD09, MMC11] estimates occupancy by measuring a variety of parameters. They collected ground truth data for 5 and 1 weeks deployments, using video camera and a user voluntary electronic tally counter to measure room occupancy respectively. In these deployments, they utilize multiple sensors to estimate occupancy; CO₂, CO, lighting, temperature, humidity, motion, and acoustics. For each parameter, they define multiple feature vectors, which are then used with several models to estimate occupancy. While this multi-sensor

approach works well for the sole purpose of occupancy estimates, this approach will not work well if combined with a ventilation strategy. They assume that ventilation will not affect occupancy estimates of the room. However, since ventilation will affect CO₂ and humidity levels and thus occupancy estimates, it is likely that ventilation rates based on occupancy estimates from this system will lead to wild fluctuations in ventilation actuation and periods of under-ventilation. In this case, ventilation is better controlled by CO₂ sensors directly even with the known calibration and response time issues [FFS06]. In essence, if CO₂ or humidity is used as sensory input, you can either control ventilation or estimate occupancy, not do both at once.

2.2 Occupant Comfort

Previous work has used building occupants as participatory sensors [RES10]. By bringing humans into the loop [EC, BTG13, JB12, Bur14, Rob14], these systems collect thermal comfort information from building occupants, and then use PMV to determine temperature setpoints to improve comfort. These works conclude that gathering sensation data such as “Feeling Too Hot” improves usability, as users are unable to determine their ideal temperature. All studies resulted in energy savings and improved satisfaction. Besides PMV, a multi-armed bandit framework [MJK13] and an optimization model [HW13] have been used to find improved temperature setpoints based on voting patterns. Although these systems provide alternative methods to translating human input to HVAC control strategy, they do not determine how data presentation and environmental feedback can be used to promote certain voting application behavior, which is the focus of our study.

[Rob14] describes operation of a comfort voting application with features sim-

ilar to our Physical and Drift mechanisms. Although algorithmic details of their control strategies are unavailable, the case study concludes that these features reduce energy consumption by 22.95%. However, due to the age of the building, and the combination of the Physical and Drift mechanisms, it is difficult to distinguish which feature led to energy savings, and how they would generalize to other buildings.

[Sch77] found that a person will act in an altruistic manner if they are shown the consequences of their actions. [PSJ07] shows that a 32% energy reduction can be achieved among dormitory residents by presenting energy usage information. However, the participants were incentivized with rewards, and 22% of participants admitted that they would not continue using the strategies after the study. [BTG13] describes the design of a metering system that presents energy consumption data and zone temperature to occupants, and energy traces over a 10 day period show that a 5% energy savings was gained when the users were given control. Providing additional energy-saving tips caused a negligible change in energy consumption. [HHP00] finds that informing people of their energy usage influences their behavior, but not always predictably. Similar results are achieved in residential electricity consumption feedback systems [ELK13,PKS14]. These studies make it clear that providing this information to participants can cause a change in behavior, but none suggest how this information might be used in an application where the improvement in energy efficiency also presents a trade-off with the voter's satisfaction and comfort. In our work, we wish to determine this relationship, and find how it can be used to create purpose-built applications for either energy efficiency or occupant comfort.

In [GCG08], social norms techniques were used to increase participation in a program at a hotel for towel reuse. They found that telling hotel guests about the

percentage of hotel visitors that had also participated in the program led to increased participation. They concluded that reporting building-level participation in the displayed statistic is ideal to draw new users into the study. In [NSC08], it is shown that although people may not believe that the actions of others will motivate them to save energy, the opposite holds true. The behavior of peers was seen to cause a large amount of savings among those users. In our study, we see if this effect holds true in a thermal comfort application when we inform the user about application usage by their peers.

Work done in [SBK11] allows homes to be efficiently heated using occupancy sensing and prediction, replacing user-programmed thermostats. Similarly, the authors of [YNF14] inform the design of future heating and cooling systems by investigating users' experiences with the Nest learning Thermostat, a commercially available smart home device. They create a set of design implications for Eco-Interaction, the design of features and human-system interactions with the goal of saving energy.

In [CFH14], a drift control strategy was deployed in the campus housing of 8 University undergraduates for a period of ~ 18 days. The authors learned that some occupants will learn to adopt new thermal regulation strategies (i.e. clothing changes, heating the room in advance) while others will struggle to maintain comfort. Changes in activation of the zones' radiators suggest energy savings up to 42.2%. Despite the similarity in the drift control mechanism, it is difficult to relate the results found in this study to ours, largely due to environmental/study conditions. For instance, the University students' schedules were not known, making it difficult to know what savings were due to the room being empty. In comparison, the participants of our study have very structured hours, and rely on comfort in their space to remain productive. In our work, we wish to determine

if this drift strategy can be used in a commercial building to save energy while still maintaining high occupant satisfaction.

Research done in [YAL14] investigates the use of different feedback mechanisms to improve the energy efficiency of an occupant’s office space by reminding/enabling the occupant to disable unnecessary equipment (lights, computer, etc.) when the user is not in the office. In our work, we wish to reduce the energy consumption of the HVAC system while it conditions the user’s space. For this reason, we must leverage energy savings against occupant comfort. Additional work [ZYL13] claims that occupant activity prediction based on real-time plug energy load data can be used to control plug load devices and HVAC system, but no evidence or experimental results are presented to support these claims.

2.3 Building Modeling

[KMB13] explores the use of MPC in a multi-zone building. In this paper a model is developed and evaluated by simulation over a 24 hour period. This paper has flexible temperature bounds set for a 6am to 6pm workday with no consideration of occupancy. [BC14] uses simulation in order to demonstrate that real-time occupancy can provide additional savings. In our paper, predicted occupancy is used in a building to show that it savings can be achieved. Further work is done in a longer period by [MKD12] which includes load balancing based on the chiller system used in their building. This paper does not take occupancy prediction into consideration.

Work has also been done to a small extent to use occupancy prediction with MPC in [GIB12]. This paper’s occupancy prediction has two main methods, the first being a reactive method where there is no occupancy prediction and the cur-

rent occupancy is said to be constant. The second method has the ground truth for the entirety of the MPC. The reactive method assumes whatever occupancy is true at the moment will hold true for the remainder the model. Our paper takes into consideration of occupancy patterns to allow for preconditioning and minimize conflicts with comfort constraints while saving energy.

In addition to traditional MPC models, work has been done in [MB] [PVR13] [PMV13] to build a Stochastic Model Predictive Control (SMPC). These efforts focus on uncertainty of occupancy and thermal load predictions and minimizes the amount the temperature falls outside the comfort constraints while optimizing the system as a whole. None of the systems do occupancy detection and only consider them as part of the load of the system. With the addition of occupancy detection, greater savings can be achieved with flexible minimum ventilation.

Work has also been done to predict a buildings occupancy through out the day. [MCM12] uses different machine learning strategies and evaluates the accuracy of each method. Based on this information they are able turn off the HVAC when it is not in use. The control strategy is not specified and there is no discussion on the comfort of the occupants used in their strategy. [ECC] uses a Markov Chain to predict occupancy of the day and the result of their savings are from lowering ventilation due to occupancy and changing the set point bounds in the building. Their system switches from occupied to unoccupied strategies if a zone is predicted to be occupied for a threshold value. This scheme is naive in that it does not use the dynamics of the room in order to ensure that the room is properly condition when an occupant enters. In our system we use MPC to find an optimal solution that can occur with any occupant patterns.

CHAPTER 3

ThermoSense: Occupancy Thermal Based Sensing for HVAC Control

The primary goal of every HVAC system is to keep occupants comfortable with the secondary goal of minimizing energy consumption. In this chapter, we will focus on reducing energy consumption in a buildings by intelligently controlling the HVAC using the knowledge of the number of occupants within each room. We also take into consideration how our models affect the ability to keep each zone's temperature within the zone's target temperature range when a room is occupied. One method of reducing consumption is to condition based on actual usage. Rooms are often conditioned assuming constant maximum occupancy. This leads to heating or cooling empty rooms. In addition, spaces only partially occupied are over-ventilated leading to loss of conditioned air and thus thermal energy.

There are two different parts of HVAC systems usage that are affected by occupancy; temperature and ventilation. Temperature control is dependent on whether a space is occupied. In this case, a passive infrared (PIR) or ultrasonic sensor is sufficient to have a binary indication if a space is occupied. Binary sensing is common for lighting control and has also been used for temperature control [ABD11, LSS10] applications. However, ventilation, used to control CO₂ levels and indoor pollutants, depends on the number of people oc-

copying a space. One method is to simply regulate CO₂ levels directly with a CO₂ sensors. However, these systems are slow to respond and are prone to calibration errors [FFS06]. Since standards exist for ventilating based on room occupancy [ASH07], sensors that measure occupancy can be used to regulate ventilation. Cameras have been used for measuring occupancy [TS, EAC] but have several drawbacks. They are sensitive to sudden lighting and background changes. Camera placement is also an issue. Cameras placed in office spaces raise privacy concerns. While cameras can be placed in public hallways and function as an “optical” turnstile, this strategy is prone to cumulative error [EAC]. If an optical turnstile misses even a single person entering/exiting a space, this error is propagated until another offsetting error occurs or some other mechanism, such as assuming the room is empty at 4am, is used to remove the cumulative error. For example, if the last person in an office leaves at 6pm and this transition is missed, then the occupancy will remain erroneously at one.

In this chapter, we develop ThermoSense, an occupancy monitoring system that utilizes thermal based sensing and PIR sensors. We develop a novel low-power multi-sensor node for measuring occupancy utilizing a thermal sensor array combined with a PIR sensor. The thermal sensor array used is able to measure temperatures in an 8x8 grid pattern within an 2.5mx2.5m area. We show it is possible to use these temperature readings in order to determine how many people are within the space. Unlike the CO₂ sensor, the thermal array can measure occupancy in near real-time. The thermal array is also not sensitive to optical issues, such as lighting or background changes. By adding a PIR sensor, we increase accuracy of detecting empty spaces and the overall accuracy of the platform. The PIR sensor is also used to reduce the node’s power consumption by triggering the mote and thermal array only when someone is present. We test this new platform with a 17-node deployment covering 10 building areas totaling

2,100 sq. ft. for a period of three weeks. Using this data, we tested four different usage based conditioning strategies and analyze the energy usage; we show that 25% annual energy savings are possible with occupant based conditioning strategies. In this chapter, we contribute the following:

- We developed a novel multi-sensor platform for estimating occupancy utilizing a thermal sensor array and a PIR sensor. We performed a full system power consumption analysis and tested a 17-node deployment over a 3 weeks, in 10 HVAC conditioning areas, covering 2,100 sq.ft.

- We developed a new occupancy regression process using the sensor data, which includes thermal map background update, feature extraction, occupancy regression and post-processing filtering. We tested three regression, including K-Nearest Neighbors (KNN), Artificial Neural Networks (ANN), and linear regression (LR). Using this process, we showed that these types of sensors are capable of estimating occupancy with an RMSE of *only* ≈ 0.35 persons.

- We tested four different strategies for HVAC conditioning and show how different sensing and actuation strategies affect energy consumption and occupant comfort. We showed that by using this system for conditioning usage based control of temperature and ventilation we can save 25% energy annually while maintaining occupant comfort.

3.1 ThermoSense

In this section we discuss ThermoSense, a wireless sensor network of nodes that can measure occupancy using a combination of thermal readings and PIR. Figure 3.1 shows the ThermoSense node used for the sensor network.

In order to have an effective wireless occupant measurement system suitable

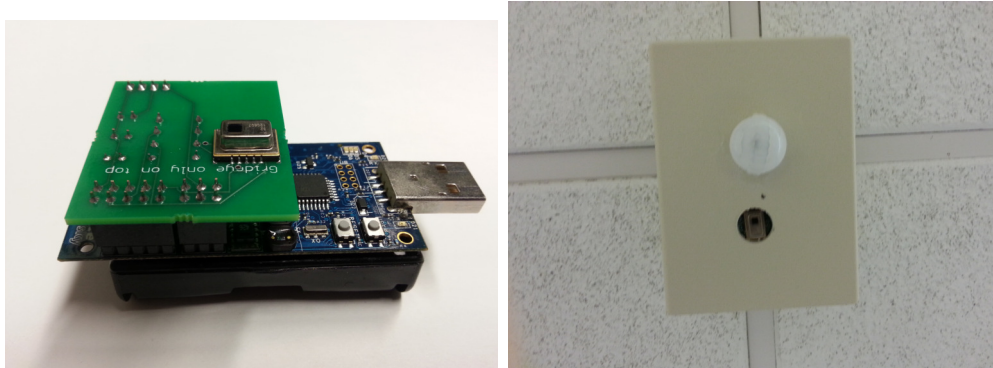


Figure 3.1: Grid-Eye attached to a Tmote (left). Enclosure containing both the Grid-Eye and PIR (right).

for HVAC control, several design considerations must be examined. While wired system for binary occupancy sensing exists, these systems are costly to install and can be difficult to retrofit into older buildings; for our system, we want an easy to deploy wireless system. Since it is wireless, power consumption is a significant issue. Both the mote and sensors must be low-powered and run in a power efficient manner. For our platform, we make use of a PIR for two reasons. The first is that accurate detection of empty rooms is critical in order to capitalize on potential savings. The second is that the PIR can also be used to sleep components in the system when sensing and communication are not necessary. While the PIR is able to give an accurate and reliable binary indication of occupancy, we need another sensor capable of determining how many occupants are in an area. Since occupants are typically warmer than the surroundings, a thermal sensor array, capable of measuring multiple temperatures with an area, is viable method for detecting occupancy. Figure 3.2 shows an above view of the area, where each square of the grid is a point of measurement of the thermal sensor array. The shaded red squares shows the points in the grid with higher temperatures, which in this case is the location of the occupant. Multiple occupants can

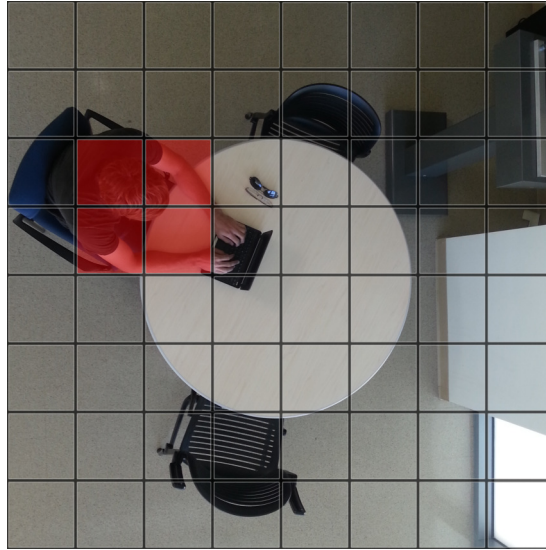


Figure 3.2: 8x8 thermal array sensing an occupant.

be distinguished by higher density of warmer temperatures.

3.1.1 Hardware

Each ThermoSense node contains a Tmote Sky, PIR, and Grid-Eye Sensor. The Tmote Sky [tmo], produced by MoteIV, has a 8MHz TI-MSP430 micro-controller with 48k Flash storage and 10k RAM. The Tmote communicates using a Chipcon CC2420 radio. The PIR sensor is connected to the Tmote's digital input/output pin while the Grid-Eye is connected to the I2C clock and data pins. We developed a board in order to connect both the Grid-Eye and PIR sensor with the Tmote.

The PIR used in the nodes is the PaPIR EKMB VZ series developed by Panasonic. The PIR is able to detect motion up to 12m with a detection a viewing angle of $102^{\circ} \times 92^{\circ}$ horizontal by vertically respectively [PaP]. The infrared thermal array used to detect actual occupancy is the Grid-Eye sensor made by Panasonic [Gri], priced at \$31. This sensor measures 64 temperatures in an 8x8 grid where the physical size of this grid is determined by the distance the

Components Used	Energy Usage.
Mote Only	4.72 mA
Mote + PIR	4.73 mA
Mote + PIR + Grid-Eye	4.76 mA
Mote + PIR + Grid-Eye + Radio	4.82 mA

Table 3.1: Energy Usage for independent components.

sensor is to the target surface. We found the Grid-Eye to be able to sense within a 2.5mx2.5m square when placed at a height of 3m. The Grid-Eye measures temperatures from -20C° to 80C° with an accuracy of $\pm 2.5\text{C}^\circ$ and can sample 10 times per second (each sample contains 64 temperature values).

3.1.2 Power Consumption

We used two 3000 mAh lithium batteries for each node in our deployment. We found battery life of ThermoSense node to be over three weeks sampling once every five seconds. For a more detail analysis of the consumption, we measure each component of the platform separately using an oscilloscope. Table 3.1 summarizes the results. The difference in energy between using PIR and Grid-Eye together as compared to the PIR only is small (0.03 mA). Adding the radio with the Grid-Eye and PIR only increases the current use by 0.06 mA. Since the samples were only sent once every 5 seconds, the energy use of the radio is small. Figure 3.3 shows the lifetime of the system at different duty cycles for different battery capacities. Since we have access to 80 Ah batteries that fit in our enclosure, we can estimate how long this system would last with larger capacity batteries. We can see even in the worse case, an 80 Ah battery would last longer than 1 year. If we were to sleep various components of the mote and if we were

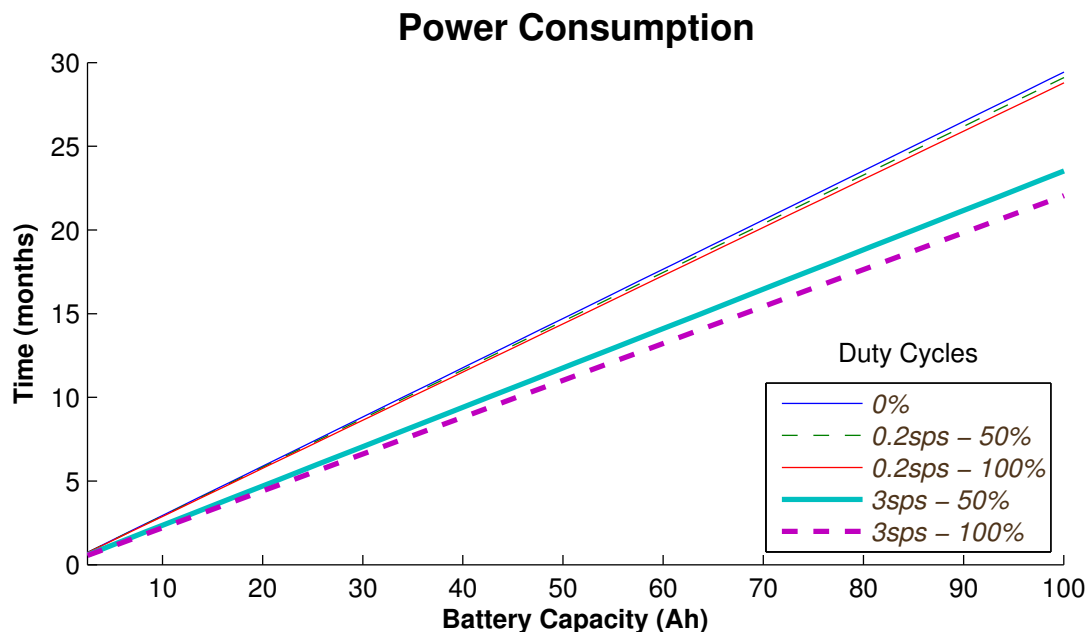


Figure 3.3: Energy usage for different duty cycles.

to do smart local data processing to transmit only deviations from a model, we could even further reduce power consumption and extend system lifetime. These optimizations are left for future work.

3.2 Occupancy Regression

In this section, we develop the process for occupancy regression in order to estimate occupancy from the ThermoSense node. Figure 3.4 shows an overview of this process. Information from the PIR and thermal sensors is used to update and maintain background levels of the thermal map. The current values along with the current background are used to create features vectors. These feature vectors are inputted into a regression model, which produces a “raw” occupancy estimate. A filter is then applied to the “raw” occupancy along with past occupancy estimates to produce the final occupancy estimate.

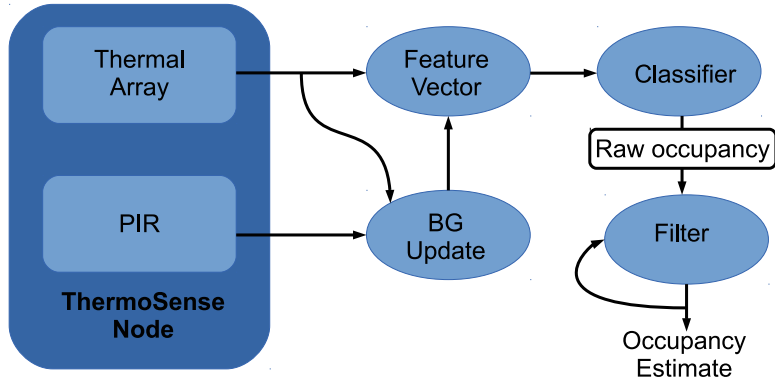


Figure 3.4: Occupancy Regression Process

In the following sections, we develop and analyze each component of the occupancy regression process and examine the overall performance of the resulting occupancy estimates. We first examine the performance of the PIR sensor, and examine in particular the ability of the sensor to detect empty rooms. We next describe the process used to maintain the thermal background using the thermal array and PIR sensor. Then we describe how the background and current thermal map is used to create the feature vectors used for the occupancy regression. We examine three different regression methods; K-nearest-neighbors (KNN), linear regression (LR), and artificial neural networks (ANN). We then discuss the use of the filter on the “raw” occupancy output of the models. Finally, we examine how the final output of the process performs by comparing the output to ground truth data collected over a 24-hour period of time from manually processed video feeds.

3.2.1 PIR Sensor Input

In our ThermoSense nodes, the PIR is used to detect if a room is currently occupied or unoccupied. The nature of the PIRs only allows motion detection, but people are not constantly moving when they are occupying a room. The PIR signal constantly fluctuates when motion is detected, and this requires a smoothing method to process it. Therefore, we smooth the raw PIR values over a period of 8 minutes. This period was found by evaluating multiple different time windows and it is sufficient time to allow someone to remain inactive for a short time while still being able to detect that the room is being used. An 8 minute period was also found to return the least number of false positives. These false positives can occur when a person momentarily enters a room (e.g. a janitor) or positives that continue to be returned after a person has left a room (e.g. motion activity before leaving). This smoothing compared to ground truth can be seen in Figure 3.5.

Due to low number of false negatives, seen in Figure 3.6, the PIR can be used reliably as an unoccupied indicator. This can be used to override any predictions that would normally be made by our thermal array sensor. Once a room has been established as occupied by the PIR, additional models can be used to further evaluate the actual occupancy of a room.

3.2.2 Thermal Background

Since occupants are typically by far the warmest objects in a conditioned room, the thermal array can be used to detect occupants within a space. However, in order to distinguish between passive warm objects such as computers or refrigerators and humans, we maintain a thermal background map. If the PIR sensor indicates the room is empty, then this information can be used to determine the

thermal background of the space. As the background can change over time, this background is continuously updated. In addition to maintaining the background, the standard deviation of each grid position is also saved; this standard deviation is used in the following section as a thresholding parameter for distinguishing significantly warm grid components. Algorithm 1 defines how our current background and standard deviation for each pixel for this background is maintained. a If the PIR has detected no movement for the 15 previous minutes, the background is updated using an exponential weighted moving average (EWMA) and the standard deviation is updated for each grid component. We found 15 minutes worked best, as it is unlikely an occupant remains motionless for 15 minutes. This threshold is also commonly used for PIR based lighting control [NLP98]. However, if an occupant occupies a space for a significant period, it is possible that the background changes during this period. To adjust the background while the space is occupied, we chose a few grid points with the lowest temperatures as our scaling components. The points with the lowest temperatures are most likely unoccupied and can be used to update the old background. We divide these scale points with the old background and average them to find a multiplier we can use to update our previous background. We then multiply the scale to the old background to find out a new background. The new background is smoothed into our old background by using an EWMA but with a significantly lower weight applied.

3.2.3 Feature Vectors

We next define feature vectors that we will use as input for the regression models. While it is possible utilize all 64 values of the thermal array along with the PIR sensor, we found that this approach did not generalize well; the models would

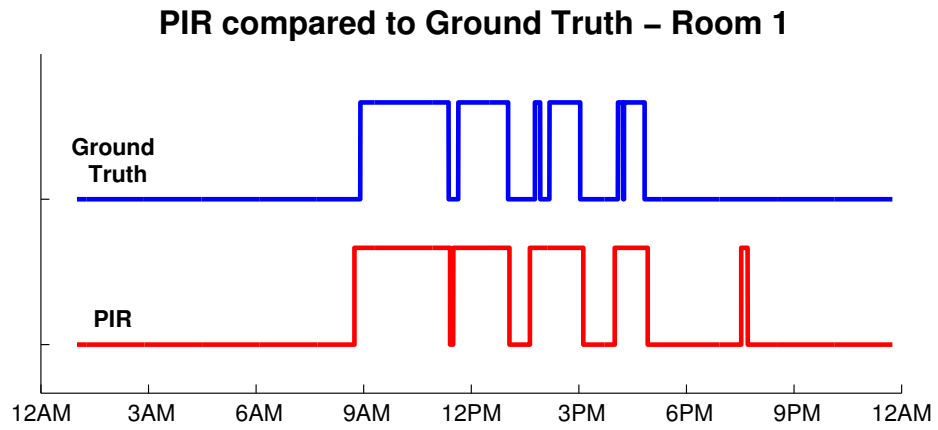


Figure 3.5: PIR compared to ground truth.

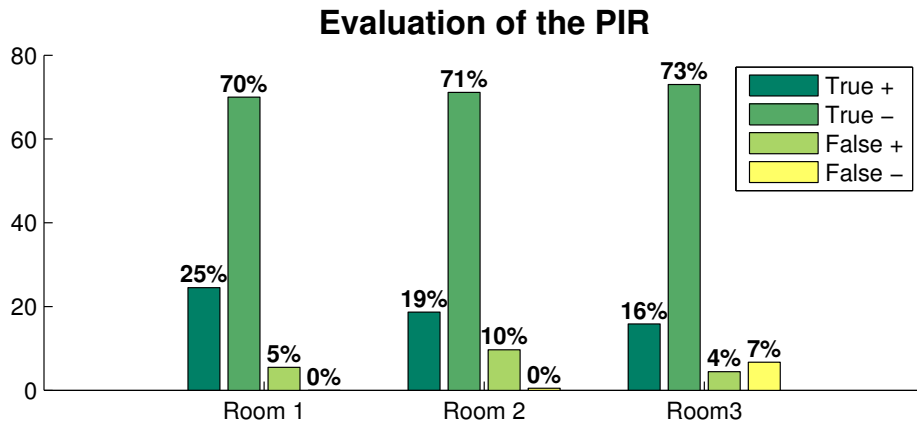


Figure 3.6: PIR evaluation of all three rooms

Algorithm 1: Background Update

$EWMA(a, x, y) \leftarrow$ EWMA with weight a .

$minTemp(f, n) \leftarrow$ indices of the n -lowest temperatures from frame, f

$frame \leftarrow$ current frame

$newBg \leftarrow$ returned updated background

$oldBg \leftarrow$ old background

$windowBG \leftarrow$ sliding window of the background

if PIR has been off for more than 15 minutes **then**

$newBg = EWMA(0.1, oldBg, frame)$

 add $frame$ to $windowBG$

 threshold $\leftarrow 3 * std(windowBG)$

else

$indices \leftarrow minTemp(frame, 5)$

for each $index$ in $indices$ **do**

$scalePx(index) \leftarrow oldBG(index) / frame(index)$

end for

$scale \leftarrow mean(scalePx)$

$scaledBg \leftarrow scale * frame$

$newBg = EWMA(0.01, oldBg, scaledBg)$

end if

not work well when applied to different areas. Instead, we use the following three features for our models; the total number of significantly warm points, the number of grouped points that are warm, and the size of the largest group of warm points. The following subsections formally defines how we extract these features. We tried a significant number of other less successful feature vectors, e.g. 64 raw values, 64 thresholded-binary values, size of all connected components, different permutations of the feature vectors, etc., but cannot discuss these due to space limitations.

All three feature vectors are based on identifying the significantly warm points on the grid. As our first step, we first create a 8x8 binary matrix representing the significantly warm points from the thermal map. This is done by taking the difference between the background and the current thermal map and applying a standard deviation based threshold to this difference. We define current thermal values, background, and standard deviation as $M_{ij} = (m_{0,0}...m_{7,7})$, $B_{ij} = (b_{0,0}...b_{7,7})$, $S_{ij} = (s_{0,0}...s_{7,7})$, respectively. An active point is defined as being three standard deviations away from the background,

$$f(i, j) = \begin{cases} \text{True,} & \text{if } M_{i,j} - B_{i,j} > 3 * S_{i,j} \\ \text{False,} & \text{if } M_{i,j} - B_{i,j} < 3 * S_{i,j} \end{cases}$$

Feature 1: Total Active Points - For our first feature, we use the total number of active points. A larger total is correlated with higher number of occupants.

Feature 2: Number of Connected Components - Our second feature is based on connected components [HT73]. Connected components is a method of identifying groups of points within a matrix that are connected to each other. A point is considered connected if it is the same value as the point diagonal or directly adjacent. A component is a group of connected points. The number of the

components, which are number of grouped warm points in our application, is correlated to the number of people occupying the area.

Feature 3: Size of Largest Component - The third feature is the size of the largest component with the grid. Multiple occupants standing close together will create one large component rather than several separate components which would results in a lower occupancy count. The size of the largest component is positively correlated with occupancy and can be used to get a more accurate count.

3.2.4 K-Nearest Neighbors

The first model we consider is K-Nearest Neighbors. Let $X = (x_0...x_n)$ be the the components of the feature vector of the current frame. $Y = (y_0...y_n)$ is each individual feature components found inside the entire training set, $Z = (Y_0...Y_m)$. We use euclidean distance to calculate the distance between feature vectors,

$$d(X, Z_j) = \sqrt{\sum_{i=0}^n (x_i - Z_{ji})^2}$$

We then find the minimum k distances from the training set of Z and collect the distances as $D = (d_0...d_k)$ with it's corresponding occupancy labels as $L = (l_0...l_k)$. Weight is applied depending on the distance to each labels to get the final predicted occupancy, P .

$$P = \sum_{i=0}^k w_i l_i, \text{ where } w_i = 1 - d_i / (\sum_{j=0}^k d_j)$$

When calculating KNN, we find the 5 nearest neighbors and the distances associated with each. The predicted occupancy is averaged with the labels asso-

ciated with the 5 closest values, with the largest weight given to values with the smallest distance.

3.2.5 Linear Regression

The next model we examine is a linear regression model. We define a linear model

$$y = \beta_A A + \beta_S S + \beta$$

where y is the estimated occupancy (predicted variable), A is the number of active pixels and S is the size of the largest component (indicator variables). β_i is the corresponding coefficient for the indicator variable i and β is a constant. While the other models also included the number of components as a parameter, for the linear model, we chose A and S by testing permutations of the feature vectors that minimized root mean squared error (RMSE) and had significant coefficients ($p < 0.05$). We also found that the model fit best when estimating positive occupancy. Thus, we train the linear model with data for the 1, 2, and 3 person cases and rely on the PIR sensor to determine if a space is empty.

Table 3.2 shows the fitted model parameters along with p values of each parameter and the F-test of the overall model. Using a $p < 0.05$ threshold, we see the p -values for each of the indicator variables is significant. We see also find that $p \approx 0$ from the F-test and $R^2 = 0.858$, indicating a good fit. Figure 3.7, shows the distribution of the residuals. The normal distribution verifies that the independent error assumption has not been violated.

	Value	p -value
β_A	0.141	2.44×10^{-187}
β_S	-0.051	1.14×10^{-30}
β	0.201	9.25×10^{-11}
F-statistic	3220	≈ 0
R^2	0.858	

Table 3.2: Parameters of linear model and fit metrics.

3.2.6 Artificial Neural Network

The final model we consider is a forward feed ANN [Mit97] using a single hidden layer of 5 perceptrons. We use a sigmoid for the hidden layer transfer function and a linear transfer function for the output layer. The same data for the 1, 2, and 3 person case was used for the ANN model. Again we use the PIR value to determine the 0 person case. We used 70% of the data for training, 15% for testing, and 15% for validation. We found $R^2 = 0.893$ and $R^2 = 0.906$ when compared with the testing set and the entire dataset respectively, suggesting the model has a strong fit.

3.2.7 Filter

The last step of the occupancy regression process is the filtering of the raw occupancy estimate. If we examined the error of the models, we found that the errors are independent and normally distributed (see Figure 3.7). Thus, we employ a 4 minute moving average filter in order to reduce error of the raw occupancy estimate produced by the regression model. We found that the 4 minute window minimized our error and was found through trial and error. Figure 3.8 compares

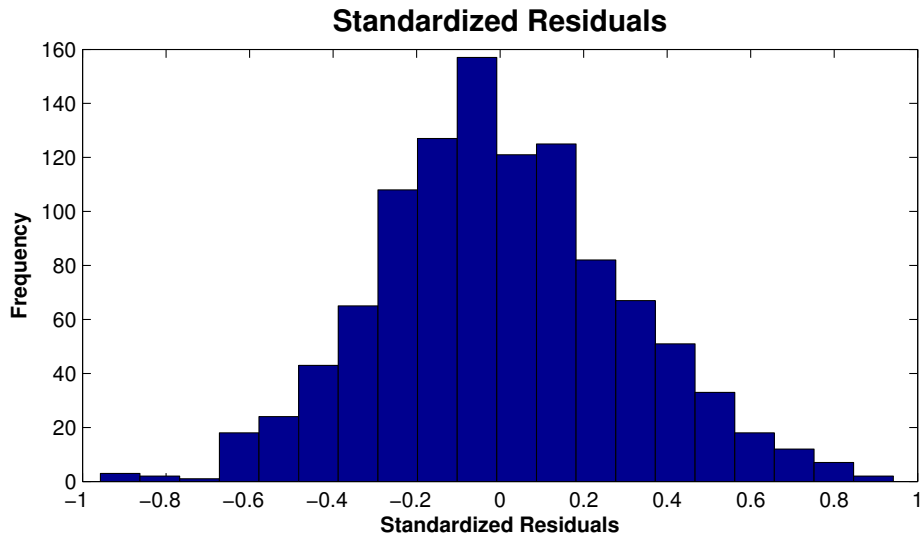


Figure 3.7: Plot of residual distribution

the raw output of the model with the filtered output. From the figure, we can see that the filter is effective at removing independent errors.

3.2.8 ThermoSense Performance

In this section, we examine the performance of the process with respect to the three models. In order to test the performance, we gathered 24 hours of ground truth data. This data was gathered by deploying a camera in a public hallway and manually counting the number of people entering and leaving the zone through out the day. For our analysis, we examined a single zone comprised of 3 separate offices. For our performance metrics, we use RMSE and normalized root mean squared error (NRMSE).

Table 3.3 summarizes the performance results. Overall KNN, performed the best. KNN had a RMSE of 0.346 people (NRMSE of 11.5%). LR and ANN had slightly higher RMSE values of 0.409 people and 0.385 people (NRMSE of

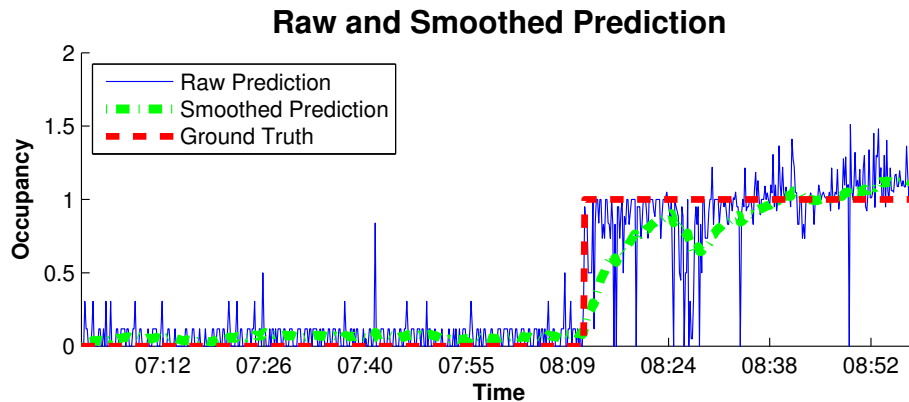


Figure 3.8: Model raw and filtered outputs.

13.6% and 12.8%) respectively. Figure 3.9 shows the performance of the models as a function of the amount of training data available. In particular, KNN is able to perform fairly well relative to the amount of data available. With only 100 training samples, KNN has a NRMSE value of 25%. Both LR and ANN have values 35% and 34% respectively when trained with the same 100 samples. LR and ANN have very similar performance between 100-900 samples. After 900 samples, we see that the ANN starts to have slightly better performance than LR. Though KNN maintained lower NRMSE overall, the difference among the different models became small as the available data increased; the maximum difference among models is only 2.1%.

3.2.9 Model Discussion

While each model had similar results with KNN having the best RMSE, there are other issues to be considered. KNN required the least amount of data in order to achieve a low NRMSE. However, Figure 3.9 shows that as more training data is available, the other models may well match or outperform KNN. Another consideration is that as the number of training data increases, the run-time per-

	KNN	LR	ANN
RMSE	0.346	0.409	0.385
NRMSE	11.5%	13.6%	12.8%

Table 3.3: Evaluation of the models used.

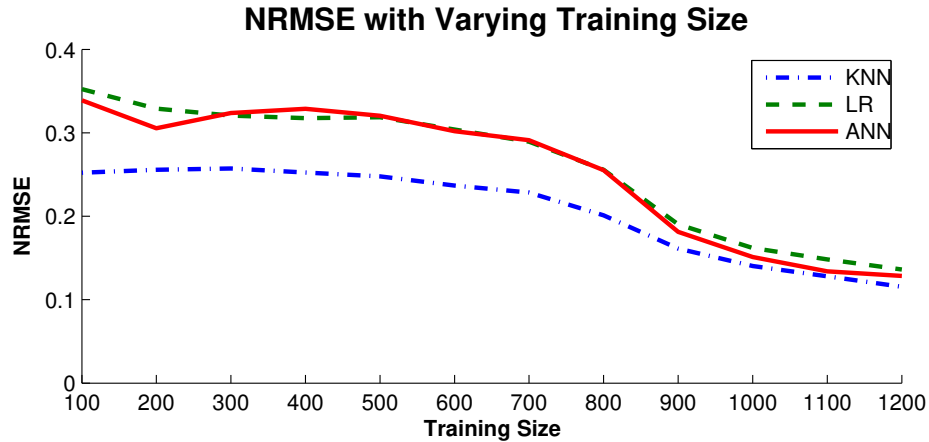


Figure 3.9: NRMSE of a ThermoSense node in a zone.

formance of KNN will decrease. While there is a possibility of running KNN on a mote, it requires storing the entire training set and iterating through the set for each new sensed sample. Thus, KNN is more suited to situations with little data exists and KNN is processed at the base-station rather than the node. Though the linear model had the lowest accuracy, there are other advantages to this approach. In particular, it can be run very efficiently on a mote; in our case, only the values of the coefficients $(\beta, \beta_A, \beta_S)$ need to be stored on the mote and new estimates are simple to calculate in real-time. A similar argument can be made for ANN; only the input and weights need to be stored on the mote and estimates are trivial to calculate in real-time. ANN can require potentially more time to train, but as this is a one time cost, ANN may be preferable over LR since ANN can handle non-linearities.

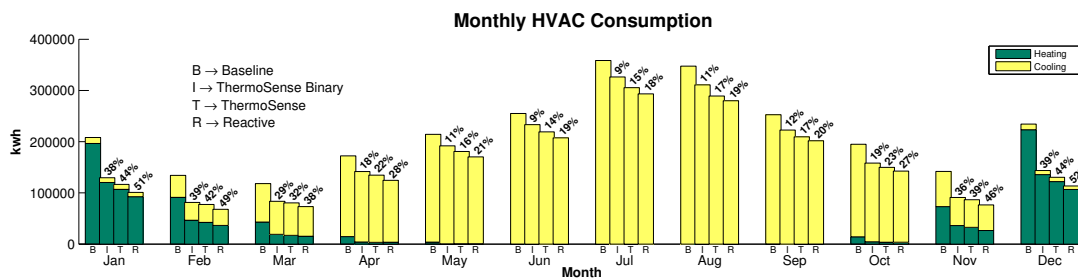


Figure 3.10: Monthly heating and cooling consumption for the different strategies.

3.3 Energy Analysis

With the occupancy estimation system in place, we next would like to use data collected from the ThermoSense system to estimate potential energy saves using different strategies. Since KNN had the best performance in terms of RMSE and NRMSE, we use this model for our energy analysis. In this section, we evaluate four strategies to reduce energy usage and compare them to a baseline strategy that operates on a static schedule. We use the OBSERVE strategy described in [ECC]. The OBSERVE method utilizes a Blended Markov Chain (BMC) that continuously updates its prediction based on the current occupancy estimate. If trained with binary data, this model can give binary predictions of occupancy. With discrete training data, the model is able to predict discrete levels of occupancy. Based on a probability threshold, the room is conditioned beforehand if the room is likely to be occupied in the coming hour. Ventilation is controlled according to ASHRAE 62.1 [ASH07]. For our analysis we use predictions discrete occupancy, binary predictions, and a purely reactive strategy using no prediction.

We trained the BMC's using 3 weeks of data collected from the ThermoSense system deployed in 8 offices, 1 lab and 1 conference room in an academic commercial building. Using the BMC, we generated 7 days of simulated occupancy. To simulate system error, we introduced error based on a normal error distribution.

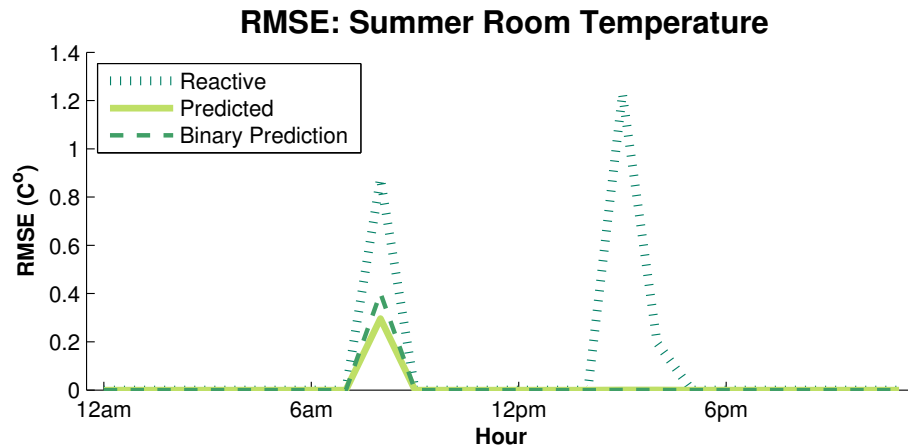


Figure 3.11: The temperature RMSE for the periods when the room was occupied during the summer.

This distribution was determined by examining the distribution of the residuals from our RMSE analysis and performing a normal fit. The frequency of these perturbations was added using an exponential distribution. We found that the duration between the errors were exponentially distributed.

To test these strategies, we utilize EnergyPlus [EPL], which is a state-of-the-art industry standard tool for simulating buildings. With this simulator, we are able to change occupancy over time to determine how these strategies affect efficiency. By using a simulator, we are also able to test the strategies under identical weather conditions. In our simulation, we used materials similar to the actual building and sized rooms to match the deployment. Target set-points of 24°C (75.2°F) and 20°C (68°F) was used for cooling and a cooling and heating set-points respectively. Since our deployment only covered part of the entire building, we assumed the same schedules for the rest of the building in the simulation. The simulated commercial building contains offices, labs, and meeting rooms. The location of the simulation was in the central valley of California. Other locations

were not evaluated in the interest of space.

Figure 3.10 shows the monthly breakdown of the heating and cooling. All three strategies had significant energy savings over the baseline strategy. Overall, the reactive strategy had the lowest energy consumption. This strategy consumed 29.6% less than the baseline strategy annually. It saves additional energy since it does not pre-condition rooms ahead of time. However, we will see that this savings in energy comes at the cost of comfort (Section 3.4). In general, all the strategies had the largest percentage of savings during winter months (Nov - Feb) and the lowest percentage of savings during the warmer months (May - Sep). ThermoSense and ThermoSense Binary had 24.8% and 19.7% savings respectively. The binary approach consumed more energy since this strategy has a tendency to over-ventilate. Over-ventilation reduces efficiency since it increases the amount of outside air that needs to be conditioned. The greatest differences between the ThermoSense and ThermoSense Binary approaches occur during the warmest and coldest months (Jan, Dec, Jul, Aug) where ThermoSense consumed 5-6% less than ThermoSense Binary. Increased ventilation of ThermoSense binary during these months has a greater impact on efficiency than during milder shoulder seasons.

3.4 Conditioning Effectiveness

In the previous section, we show that occupant based conditioning saves a significant amount of energy. However, conditioning effectiveness also needs to be considered.

For the temperature effectiveness, we examine a room that is only occupied at 8am, 3pm, and 4pm on Mondays and focus our analysis to the warm months

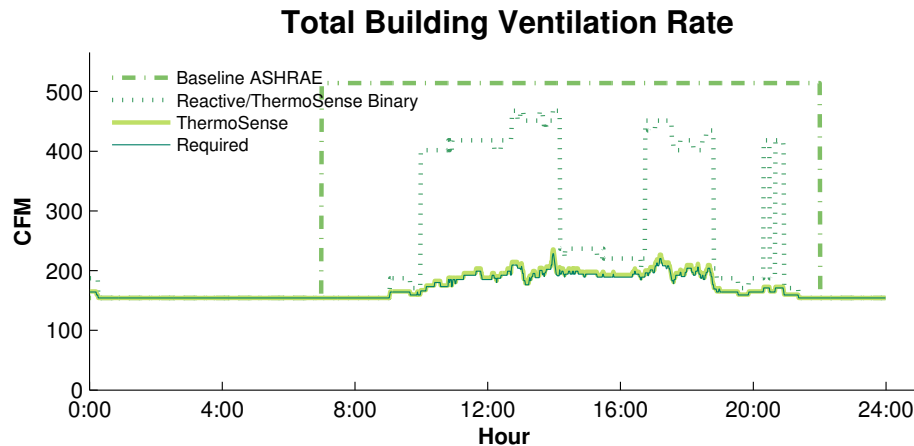


Figure 3.12: Ventilation effectiveness of the strategies.

(May - Sep). For our analysis, we examine the RMSE of the room temperature from the target temperature for each hour. Figure 3.11 shows the RMSE for each hour of the day. For un-occupied periods, we consider the RMSE to be 0. The reactive strategy had the worst overall performance. At 8am, the RMSE was $0.877C^{\circ}$ during the hour that the room was occupied. At 3pm the room had a RMSE of $1.23C^{\circ}$. This higher RMSE is due to the solar gain increasing the temperature of the room. However, at 4pm the RMSE drops to $0.197C^{\circ}$. In this case, the occupied state of the room at 3pm carried over to the 4pm period. Thus, while some energy was saved by not preconditioning, we can see that this was at the cost of thermal conditioning. Both ThermoSense prediction strategies perform substantially better than the reactive strategy. At 8am the ThermoSense and ThermoSense Binary prediction methods had RMSE values of $0.309C^{\circ}$ and $0.415C^{\circ}$ respectively. At 3pm-4pm, both prediction strategies had nearly identical RMSE values; both were close to $0.033C^{\circ}$. Again, while the ThermoSense prediction strategies consumed slightly more energy, they did not compromise thermal conditioning.

Next we examine the ventilation effectiveness of the system. For our analysis, we evaluate the ventilation of the spaces for one particular day. As previously mentioned, we use the ASHRAE 62.1 standard to determine the ventilation according to occupancy. As ventilation is regulated in real-time, the binary based strategies (reactive and ThermoSense binary prediction) have the same ventilation rates. As we cannot determine the precise occupancy with a binary occupancy estimate, we assume maximum occupancy for rooms that are occupied for ventilation. The baseline strategy assumes maximum occupancy from 7am to 10pm. Figure 3.12 shows the baseline, ThermoSense binary, and ThermoSense based strategies. We can see that the baseline strategy greatly over-ventilates. However, there is also a short period at the beginning where the baseline strategy under-ventilates; this illustrates the shortcomings of a static schedule. Both the ThermoSense binary and ThermoSense strategies were able to meet the required ventilation level. However, we can see that the ThermoSense binary strategy still over-ventilates the area a great deal; overall, this strategy over-ventilated the area by 170%. The ThermoSense binary strategy performed best when multiple rooms were empty, which can be seen during the period between 2pm and 5pm. Since only a few rooms were occupied, only a few rooms were over-ventilated. The ThermoSense based ventilation performed the best. We can see that the ventilation rate is only 3.25% more than the required rate. This is due mainly to a 10% occupancy increase added by design in order to protect against under-ventilation that could be caused by sensor errors [ECC].

3.5 Summary

In this chapter, we developed ThermoSense, an occupancy monitoring system that utilizes thermal based sensing and PIR sensors. We developed a novel low-

power multi-sensor node for measuring occupancy using a thermal sensor array combined with a PIR sensor. We showed it is possible to use these temperature readings in order to determine how many people are within the space using a novel occupancy regression process. Unlike the CO₂ sensor, the thermal array can measure occupancy in near real-time. The thermal array is also not sensitive to optical issues, such as lighting or background changes. By adding a PIR sensor, we increased accuracy of detecting empty spaces and the overall accuracy of the platform. The PIR sensor is also used to reduce the node's power consumption by triggering the mote and thermal array only when someone is present. We tested this new platform with a 17-node deployment covering 10 building conditioning areas totaling 2,100 sq. ft. for a period of three weeks and showed that ThermoSense is able to detect occupancy with a RMSE of *only* ≈ 0.35 persons. Using this data, we tested four different usage based conditioning strategies and analyzed the energy usage; we showed that 25% annual energy savings are possible with occupant based conditioning strategies.

CHAPTER 4

FORCES: Feedback and control for Occupants to Refine Comfort and Energy Savings

In the previous chapter, we addressed the issue of reducing energy cost using an occupancy regression. This however does not take into consideration if the occupants within the building are comfortable. In this chapter, we introduce a participatory voting system to allow occupants to vote based on their thermal preferences and for these votes to change the thermal conditions of their spaces. Unlike the previous chapter, this body of work's goal is to increase user satisfaction with a building's thermal condition while reducing energy consumption.

Although comfortable temperatures in commercial work environments make employees happier and more productive [FR97,SF05], maintaining ideal temperature for occupants is difficult to do correctly. Building managers are responsible for setting temperatures for spaces within buildings, but the chosen temperature setpoints are general estimates for the building and do not necessarily suit the thermal preferences of the individual occupants. A pre-study survey we conducted, completed by 61 occupants in 3 University buildings, indicated that 96% of those surveyed have had to take individual action to improve their comfort in their work space by using a personal heater or fan, adjusting clothing or using a blanket, adjusting their thermostat (only 18% of those surveyed reported an effective thermostat), relocating, etc. Importantly, 33% of those surveyed indi-

cated that they have had to avoid their work environment due to discomfort, 45% reporting that the thermal conditions of their work environment inhibits their ability to work efficiently.

This occupant discomfort is caused by imperfect temperature setpoint selection. The state-of-the-art method for choosing temperature setpoints is taken from the American Society of Heating, Refrigeration, Air-Conditioning Engineering (ASHRAE) Standard 55 [ASH04], which relies on Predicted Mean Vote (PMV). PMV uses parameters such as temperature and humidity to calculate an expected comfort level for an individual from -3 (“Too Cold”) to 3 (“Too Hot”). A key limitation of using PMV is the difficulty of ascertaining other parameter values, such as clothing level and metabolic rate. Our pre-study survey indicates that occupants prefer temperatures as low as 63°F and as high as 85°F, and so even with perfect PMV factor estimation, variation in occupant temperature preference makes PMV prone to error.

At home, an occupant can adjust a thermostat to suit his or her preference. However, as thermostats often aren’t available to employees in commercial buildings, most who want to modify the temperature in their space must contact the building manager and request an adjustment. According to our pre-study survey, only 13% of participants had contacted the building manager with satisfactory results, and they did so no more than twice a month. As only 34% of those surveyed reported satisfaction with the building HVAC system, this indicates that occupants are deterred by the adjustment process, and would prefer to bear the discomfort rather than repeatedly contact the building manager.

Work done in [EC, BTG13, JB12, Bur14, Rob14, RDS14] develop methods of collecting thermal comfort data. [EC, Rob14] continue by using this data to manage the thermal characteristics of the space to fit occupant preferences by adap-

tively conditioning the user's zone. In general, the occupants are found to be significantly more satisfied once given the ability to control building temperatures.

Although these comfort voting applications improve upon standard methods of building interaction, there is still work to be done. A comfort voting application in a commercial building must aim to achieve the primary goals of the building manager, typically either reduction of energy consumption, or maximization of occupant comfort. Our main contributions are the following:

- We develop FORCES, Feedback and control for Occupants to Refine Comfort and Energy Savings, a multi-platform application that a building occupant can use to vote on thermal comfort within a space. The application uses this information to adjust the HVAC system using two different control strategies to improve occupant comfort while trying to minimize energy use.
- We propose 5 application feedback types that use various methods of data presentation and environmental stimuli to promote specific behavior in using FORCES. In a 1-month preliminary study, we analyze how these feedback types affect user behavior for thermal comfort and show that green and physical feedback provide the best energy savings and user comfort satisfaction.
- In a longer 40-week study, these two feedback types coupled with two different control strategies are implemented across 61 participants in 3 different buildings to understand how their use affects HVAC system energy consumption and user satisfaction. We show an increased user satisfaction from 33.9% to 93.3% and a reduced energy consumption by as much as 18.99%. In addition, we find that by including a drifting control strategy, savings up to 37% can be realized without a significant reduction of occupant satisfaction.

To our knowledge, this is the first work that investigates how different feed-

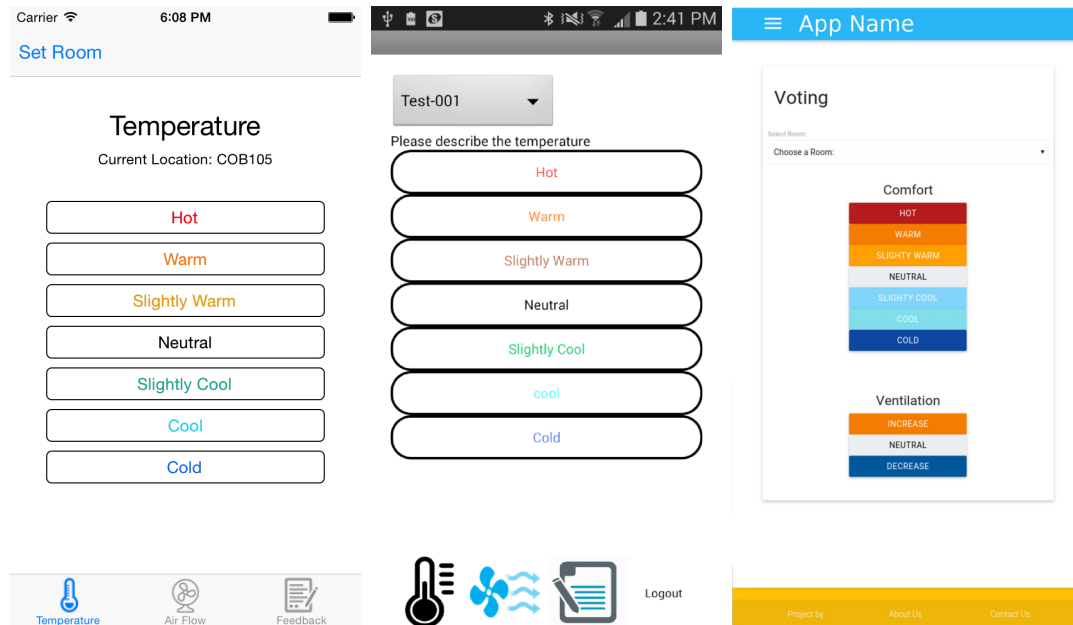


Figure 4.1: From left to right, iOS, Android, Web version

back mechanisms and control strategies can influence human decisions in a thermal comfort application controlling an HVAC system in production buildings to balance the energy/comfort tradeoff.

4.1 System Design Overview

4.1.1 System Architecture

FORCES is a comfort voting application that allows occupants to vote for their thermal preference. Based on this vote, the room's thermal conditioning is changed to better suit this preference. To facilitate user interaction with FORCES, we developed iOS, Android and Web interfaces as shown in Figure 4.1. Using strategies from [EC] we receive votes from users describing how they feel, from Cold (-3) to Hot (3). These values are defined as the Actual Mean Vote (AMV),

Name	Value
Air Speed	$0.1m/s$
Mean Radiant Temperature	$25^{\circ}C$
Humidity	30%
Metabolic rate	$70W/m^2$
Clothing Level	$0.5clo$

Table 4.1: PMV Constants

and are used to find the ideal temperature for the room by setting it equal to Fanger’s formulation of PMV [Fan70]. PMV uses parameters such as the clothing coefficient and metabolic rate that are difficult to obtain for each participant, so we use estimated parameter values obtained from [CBE] that are appropriate for an office worker adjusted by season, as shown in Table 4.1. A temperature found to be suitable according to ASHRAE Standard 55 [ASH04] is one with a PMV value between 0.5 and -0.5, where the occupant is neither too hot or cold. We can therefore solve for this comfortable temperature and set it for the room.

Figure 4.2 shows an overview of the FORCES system. When a user submits a vote for their comfort, it is sent to the FORCES API, and feedback information is sent back to the user. The collected AMVs are tallied every 5 minutes and averaged in zones with multiple users. The AMV is used by the PMV model to determine the next room temperature setpoint, which is given to the Building Management System (BMS) for actuation. The building makes the necessary adjustments to condition the room to the desired temperature. As the thermal conditions are monitored by the user and votes are used to further modify the temperature, FORCES creates a closed loop feedback system with the human as a sensor.

Setpoints are handled by the BMS, which achieves the setpoint. WebC-

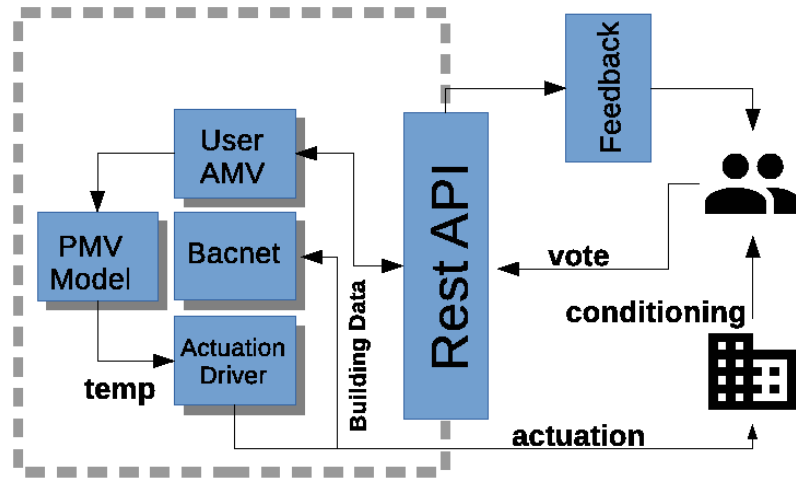


Figure 4.2: FORCES System Architecture

TRL [web] is the BMS system used on our campus. For both sensing and actuation we use sMAP [DJT10a], a RESTful API designed to manage building data. By designing an actuation driver that communicates with our building’s BMS, we leverage the control systems implemented by designers of the building instead of creating our own local control PID loops. In addition, this also allows facilities crews to have full visibility on any of our changes through the existing BMS.

To monitor the thermal state and ensure correct building operation, we develop a driver that uses the BACnet protocol [BAC99] to communicate with each sensor in a zone. The temperature and air flow rate sensors in each zone are polled at a 2 minute interval, a much finer granularity than the data collected by the BMS.

We want to make FORCES data available to applications on multiple platforms, so we design a RESTful API layer that computes the information displayed in our views. In this layer we can authenticate users, check comfort votes for validity and integrity, and uniformly present data to iOS, Android, and web users.

4.1.2 Application and Feedback Design

Application design affects user interaction in various ways. For instance, improvement of a mobile application’s user interface can improve user enjoyment. Additionally, we hypothesize that in control applications like ours where the human is in the loop, feedback provided to the user can also influence their voting patterns. Based on research discussed in *Related Work*, we choose 5 feedback mechanisms for our comfort voting application, described below in detail.

4.1.2.1 Physical Feedback

By design, the vote collection system architecture defines periods of time for votes to be collected from users before a control decision is made and actuation begins. The period chosen between actuations is typically several minutes, as the amount of time required to condition a room to temperature is large. However, with several minutes between a user casting a vote about thermal comfort and the building’s response, a user may become dissatisfied with the quality of the HVAC control.

To minimize this effect, we propose a “physical” feedback that immediately triggers a long burst of air from the HVAC system when a vote is received. This “instant gratification” provides sensory stimulation in the form of sound from the vents and an increase of air movement that assures the user that the system is changing in response to a vote. The additional activation when a vote is received may create wear and tear on the HVAC system; for this reason, we limit the feedback to just one burst per system actuation period. In our experiment, this period was set to 5 minutes. Long-term experiments are required to measure any impact of this feedback on HVAC equipment, so we leave this for future work.

4.1.2.2 Expected Action Feedback

In commercial buildings, multiple occupants are commonly required to share the same thermal zone and HVAC infrastructure due to system design. To a thermal comfort control application, this causes complications as there is no longer a one-to-one relationship between an occupant and a space. In particular, previous work [EC] showed that a shared thermal zone encourages users to exaggerate votes in order to “bias” the system towards their own preferences. One likely cause for this bias is that a user may not know what votes are being cast by other users in the shared space.

Once a user casts a vote, the expected action feedback takes a tally of all votes that have been cast in the last voting period, and then reports back to the user exactly what action the system is expected to take. To a user, this confirms that the system has received the user’s vote, and implies what votes are being made by other users in the same space. This feedback can reduce the bias observed in previous works if the system intends to react as the user wants, or could enhance the bias if the system’s planned actions conflict with the user’s vote.

4.1.2.3 Social Feedback

By informing a user about their peers’ activity, also known as the descriptive norm, it is possible to encourage a particular behavioral pattern. In [GCG08], hotel guests were found to be more willing to participate in an environmental conservation program when told that a large percentage of other hotel guests participated. Similarly, our social feedback notifies a user of the recent voting patterns of their peers, to encourage interaction with FORCES. To calculate peer participation, voting activity is determined across the entire building. The number of active users in the last week is found, and the percentage of those who

have voted in the last 24 hours is reported to the user. We calculate activity based only on active users to keep the percentage relatively high even during periods of inactivity.

4.1.2.4 Leaderboard Feedback

Another feedback type that operates on social stimuli is the leaderboard, a common form of gamification [DDK11]. The leaderboard reports a user’s “score”, and shows the user’s rank among all users with respect to this score. In our study, vote volume (i.e. total number of votes) was chosen as the score, but other metrics may be used as well. The advantage of the leaderboard is the introduction of competition into the voting platform. Users that are high on the leaderboard will be motivated to continue voting to maintain their high position, and users that are low on the leaderboard can clearly see how they compare to the top leaders. We suspect that leaderboard use helps maintain interest by inspiring competition among voters.

4.1.2.5 Green Feedback

The primary objective of a building’s HVAC system is to make occupants comfortable, so a user of an app like FORCES may vote with no thought to system energy efficiency. However, if a control change will significantly improve the building’s efficiency but minimally affect occupant comfort, the occupant may be willing to make the more sustainable vote. Green feedback informs users of the environmental implications of their voting pattern to encourage votes that are more energy-aware.

By comparing a zone’s default temperature setpoints to those used by FORCES and considering the outside temperature, we can determine how much more or

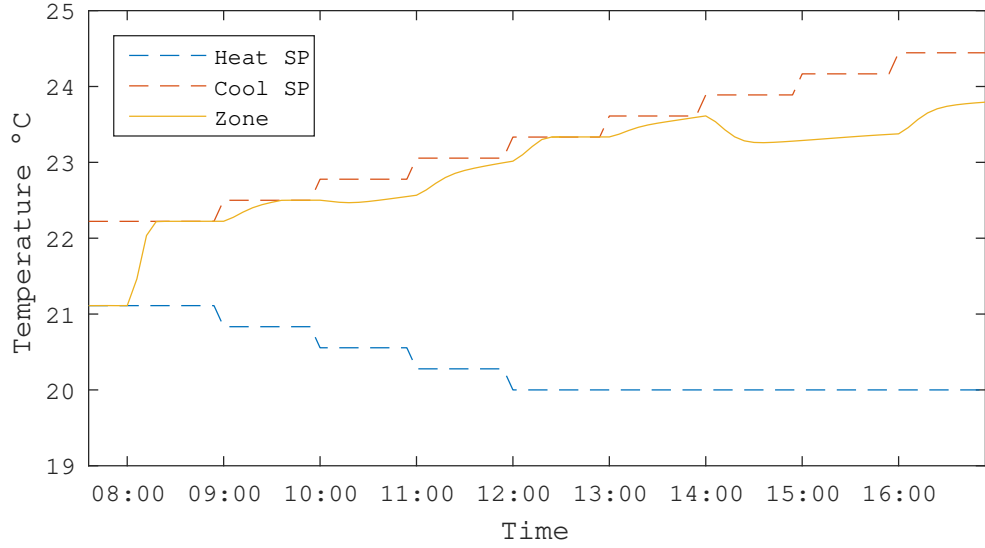


Figure 4.3: Drift strategy set-points to save energy

less energy will be consumed by responding to a vote. This difference is reported back to the user in terms of energy saved or wasted. To make the feedback easier to understand, the energy saved/wasted is converted into metrics of energy that most people will understand, such as numbers of hours a microwave could be powered or miles an electric car could travel using the saved or wasted energy.

$$\Delta T = |T_d - T_o| - |T_n - T_o| \quad (4.1)$$

$$E = \Delta T \Delta t (VR)$$

The energy savings/waste calculation is calculated as shown in Equation 4.1. We find the absolute differences between the default temperature set by campus policy T_d and the outside temperature T_o , and then between new temperature setpoint T_n and T_o . Subtracting these two values will give us how much more cooling or heating is required for that room. The energy is then calculated as the product of ΔT , the volume of the room V , the efficiency of cooling the room R ,

and time Δt . Before reporting to the user, we convert the energy savings/waste into more intuitive units by multiplying it by appropriate conversion factors. Examples of units presented to the users are: “You have saved/wasted enough energy to cook g_p pizzas”, “drive an electric car g_c miles”, etc.

4.1.2.6 Drift Control

Previous work in building control strategies [Hor99, PSP10] dictates that it is difficult to maintain user interest in a comfort voting application once thermal comfort is achieved. Similarly to [CFH14], we introduce a control strategy that allows heating/cooling setpoints of a room to drift apart. This allows temperature to “float” within these expanding bounds, allowing energy savings. Example use of such a drift strategy can be seen in Figure 4.3.

In normal usage, heating/cooling setpoints are chosen based on occupant voting patterns. However, if these bounds are held unnecessarily tightly, extra energy will be consumed. We imagine drift providing the most benefit in two situations. The first is when outside temperature is comfortable, so minimal ventilation will be sufficient to maintain thermal comfort. The second is when occupants only need temporary conditioning, for example while they are cooling off from walking inside on a hot day, or if they come in and out of the office and don’t need their room conditioned the whole time.

In our experiments, we allow the bounds to start drifting apart 30 minutes after the occupants’ last vote, and the drift moves each bound apart by $1/2$ °F every 30 minutes. In practice, the bounds are reset to ± 1 °F from the last voted temperature each morning, and unless votes occur will be allowed to float to the farthest possible bounds of 66°F and 76°F, as chosen by the building managers.

	Male	Female	Declined		
Gender	30.5%	66.1%	3.4%		

	18-29	30-39	40-49	50+	Declined
Age	33.9%	22.0%	28.8%	13.6%	1.7%

Table 4.2: Demographics of Study Population

4.2 Experimental Setup

In this section we explain the participant recruitment process, the demographics of the recruited participants, the buildings used in the study, and their installed HVAC systems. Our preliminary study examines the effect of the 5 proposed feedback systems on participant satisfaction and energy consumption across a 4-week period. The results are then used to narrow down on the two most promising feedbacks for a long-term study. In total, our studies include 61 participants over a span of 40 weeks, with more than 1300 comfort votes collected.

4.2.1 Population Description

To discern differences in voting patterns due to feedback modification, FORCES was deployed across a diverse population. The majority of the FORCES participants are full-time employees in administrative offices or research labs, with 76% working 30+ hours/week. 71% of those included in the study share their work space with one or more people. Interestingly, our study is predominantly female, aligned with the current population that occupies the building. Additional population demographics can be found in Table 4.2.

4.2.2 Recruitment Process

The study begins for each participant upon completion of a skill session with one of the project’s researchers. The skill session allows us to complete four objectives:

- 1) The participant reviews and signs the Institutional Review Board (IRB) consent form.
- 2) The participant completes a pre-study survey, providing demographics, workspace information, and initial comfort in the space. The participant will complete a post-study survey later for comparison.
- 3) The researcher helps install the application on iOS, android, or bookmark the web-voting page based on the participant’s preference. Application use is explained to the participant, describing feedback in a neutral way.
- 4) The participant was provided login credentials as well as contact information for the researchers, in case an issue was encountered.

The FORCES system operates autonomously, with manual intervention required only if a fault in the HVAC system were to occur. Building managers love the FORCES system as it reduces the need for frequent requests for temperature adjustment, reducing their workload and keeping building users more satisfied.

4.2.3 Feedback Assignment

In the preliminary study, we sequentially assigned feedback types as we recruited participants with the exception of physical feedback. As the physical feedback of one user’s vote would alter the thermal sensation of another, we chose to specifically assign physical feedback only to participants that had their own thermal zones. As the remaining feedback methods were assigned semi-randomly, partic-

Feedback Type	Baseline Energy (KWH/day)	Feedback Energy (KWH/day)	Percentage Incr.	Projected Cost Incr. (USD)
Social	83.38	89.23	7.01 %	\$17,584.74
Green	28.93	30.06	3.90 %	\$9,782.48
Physical	13.36	14.26	6.76 %	\$16,973.22
Expected	81.03	87.29	7.73 %	\$19,393.60
Leaderboard	19.05	20.98	10.12 %	\$25,389.26

Table 4.3: Energy Cost per Feedback - Preliminary

ipants that shared a thermal zone often received different feedback types. In this manner, we get a nearly-equal number of participants in each feedback type, and a fairly random spatial distribution.

At the end of the preliminary study, we found that having different feedback methods within a single HVAC zone makes energy analysis of the feedback types difficult. In the primary study, we corrected this by assigning feedback randomly across the HVAC zones, but assigning all users within a zone to the same feedback condition.

4.2.4 Building Description

FORCES was activated across three buildings, covering a total of 2 open receptionist areas, 33 offices, 8 research labs, and 14 cubicles. Most of the offices house one or two participants, while the cubicle areas and research labs can hold as many as 6 and 10, respectively. In BLD2, a terminal reheat system is present with thermafusers, allowing zones with a shared variable air volume (VAV) box to receive individual control (excluding cubicle areas, which must share control). In BLD1 and BLD3, thermafusers are not available, so in some cases individual control is not possible, even in offices.

4.2.5 Energy Calculation

Our campus has a central cooling/heating plant which stores chilled/heated water that is distributed to our buildings, which use single duct terminal reheat HVAC systems. We use a model of energy usage from [KB11], which calculates energy usage based on air temperature differentials through the cooling/heating coils and the fan electricity used to move the air. Here we describe how we follow [KB11] for use in our buildings.

The outside and circulated inside air are mixed in the economizer to T_m , the “mixed air temperature”. The Air Handler Unit (AHU) cooling coil then cools T_m to the supplied temperature T_s , to be pushed to each room for conditioning. The cost of cooling for a particular zone is computed as the mass flow of air to the zone \dot{m}_z (A fraction of the total \dot{m}_z from the AHU) multiplied by the change of temperature. A massive fan in the AHU provides air for the a group of zones. A cooling coefficient, η_c , is applied to the result as shown in Equation 4.2

$$P_c = \eta_c(T_m - T_s)\dot{m}_z \quad (4.2)$$

Once past the cooling coil, the temperature is increased by the zone’s heating coil from T_s to the discharge temperature, T_d . The power for heating is therefore the increase in temperature times the mass flow for the zone and a heating coefficient, η_h as shown in Equation 4.3.

$$P_h = \eta_h(T_s - T_d)\dot{m}_z \quad (4.3)$$

The final addition to total power use is that of the air supply fan. The power is the square of the total mass flow, m_t , multiplied by fan efficiency coefficient κ .

$$\Delta P_f = \kappa \dot{m}_t^2 \quad (4.4)$$

The AHU supplies air to multiple VAVs and thermafusers, so to separate energy cost per zone we calculate the fraction of total air used for each zone. The cooling, heating, and fan power is used to find the total power to condition the zone, P_z as follows:

$$P_z = P_h + (P_f + P_c)m_z/m_t \quad (4.5)$$

To determine FORCES’s effect on energy use, we compare each room’s energy use during the study with the *same* room’s energy use before the study, for each individual day. To minimize error due to weather variation, the pre-study day used for comparison is chosen such that its weather profile is as similar as possible to the study day. This is done by minimizing the root mean squared error between hourly outside-air temperature data trends of the study and pre-study day. As the buildings used in our experiments have never been repurposed and the employees that occupy the spaces are stable, we expect behavioral variations to be negligible in this comparison. Furthermore we only analyze weekdays, as HVAC is turned off during the weekends.

As each participant is separated into different feedback types in the preliminary study, it is difficult to map a feedback to a room’s energy usage. We select rooms where more than 80% of votes have come from a single feedback group. In addition, we only look into zones within the same building, BLD2, to prevent bias due to differences between VAV systems. In the primary study, this is not necessary, as all participants in a shared zone are assigned the same feedback type, as discussed.

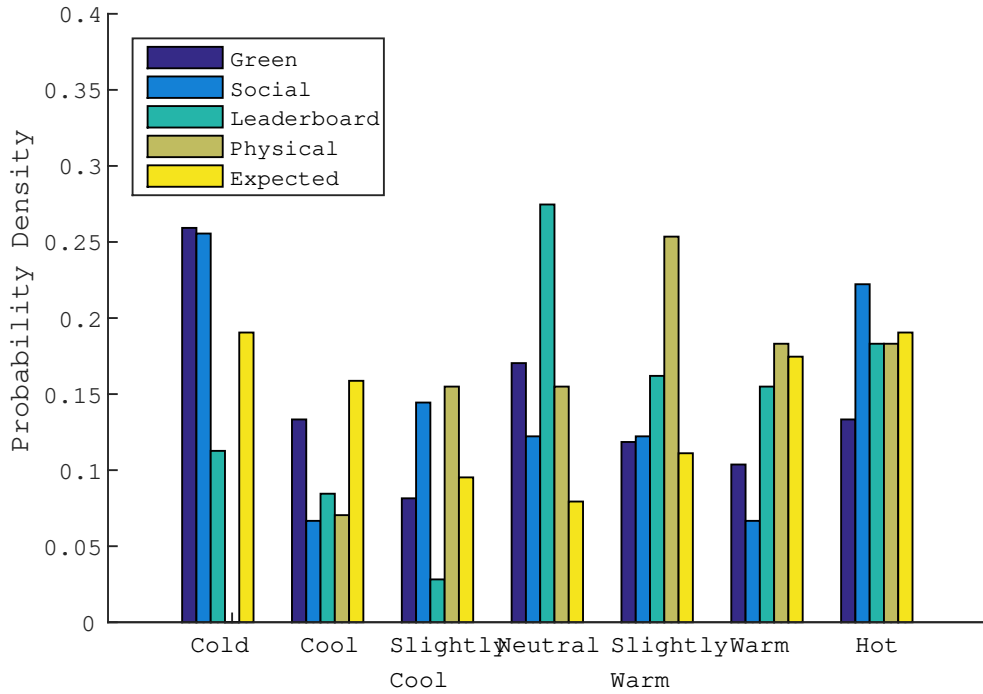


Figure 4.4: PMV vote distribution for each feedback type

4.3 Results

4.3.1 Preliminary Study

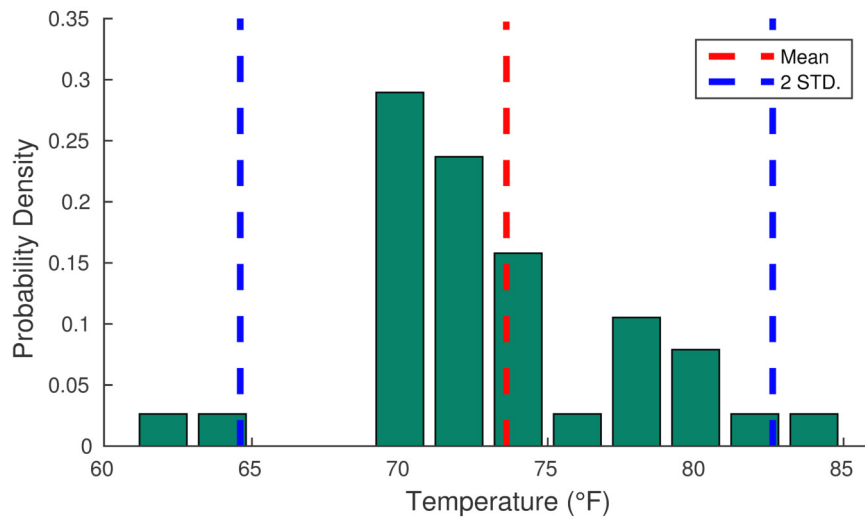
The preliminary study deployed all 5 feedback types to narrow down the ones that showed the most promise with respect to energy efficiency and user satisfaction. As this experiment was to compare feedback types only, the drift control strategy is not evaluated here.

In Figure 4.4 we see the distribution of votes along the PMV scale for each feedback tested in our preliminary study. On the x axis, the PMV range is shown from “Too Cold” (-3) to “Too Hot” (3). On the y axis, the fraction of votes with that value is shown, such that the sum of a feedback type across all comfort levels is equal to 1. The distribution of votes gives intuition into the goal of the

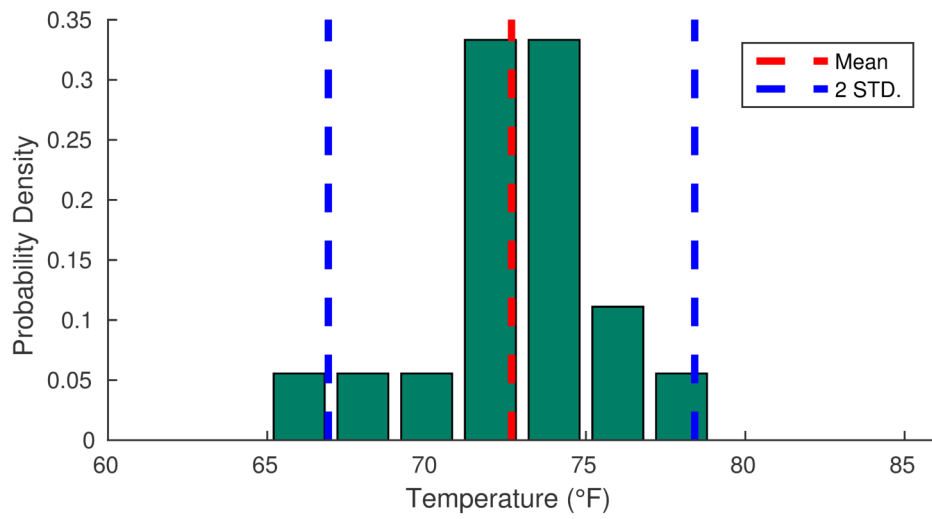
users. For instance, leaderboard and physical have the majority of votes on the right of the scale, indicating a preference for cooler temperatures. Due to the hot weather during the study, providing cooler temperatures will result in increased energy usage. However, the vote distribution of users with the green feedback is greater on the lower end of the scale, with 47.41% of participants voting below neutral, 35.56% above neutral, and 17.04% voting for neutral comfort. As each vote below neutral is a request for warmer air, it shows that users were willing to increase the temperature, decreasing energy usage.

Interesting artifacts can be found in other feedback vote distributions, as well. Leaderboard users, for instance, generate 1.61x more neutral votes than the feedback with the second highest neutral votes (green). This indicates a strategy for users to inflate their rank on the leaderboard using votes that don't greatly affect the temperature. As the Leaderboard's "gamified" interface encourages its users to provide more votes (even neutral ones), the FORCES system will have explicit affirmation that the user is comfortable (neutral), helping FORCES to establish and maintain comfortable temperatures. In contrast, with other feedback methods, FORCES must make an implicit assumption that the user is comfortable based on the absence of votes. Physical feedback is designed to provide an immediate impression to the user that the HVAC system is working. However, the application provides no incentives to reduce energy consumption. This is shown in the figure, as 61.97% of the votes submitted by users with physical feedback are above neutral, requesting cooler air at the cost of more energy. Both social and expected feedback can be seen to have a higher concentration of votes on the extremes and a reduction near neutral. This is expected, as somebody in a comfortable zone feels less need to vote.

Using the procedure discussed in *Calculating Energy*, we analyzed the energy



(a)



(b)

Figure 4.5: User temperature preference as indicated by Survey (top) and Voting patterns (bottom)

Feedback	Useful	Neutral	Not Useful
Green	33.33	33.33	33.33
Social	60.00	20.00	20.00
Leaderboard	50.00	50.00	0.00
Physical	100.00	0.00	0.00
Expected	100.00	0.00	0.00

Table 4.4: Survey: Did the user find the app useful?

efficiency of the 5 deployed feedback types over the one-month study. Table 4.3 reports the energy usage, as well as the projected cost if FORCES were installed on an entire building. As shown, the energy consumption of all feedback types increased over the baseline comparison point. The variations in baseline energy shown in Table 4.3 are caused by the differences in square footage of the spaces, number of occupants, thermal load, solar gain, and other factors. Although many of these variations are unavoidable, we don't compare these rooms to each other, but to themselves with and without the FORCES system. In this way, we can use the feedbacks' relative performance to choose the ones that performed most efficiently. In particular, these results show that Green feedback led to the least energy usage by a fair margin, with Physical and Social as runners-up.

In addition to the energy analysis performed on the 5 feedback types, we also investigate what the building occupants thought of the app's usefulness by having them complete a survey at the end of the study. As shown in Table 4.4, both Physical and Expected feedback methods were found to be useful by unanimous vote. Due to the relatively low energy usage of Green and Physical feedbacks and the usefulness of Physical feedback, we think these two are best suited for the long-term study.

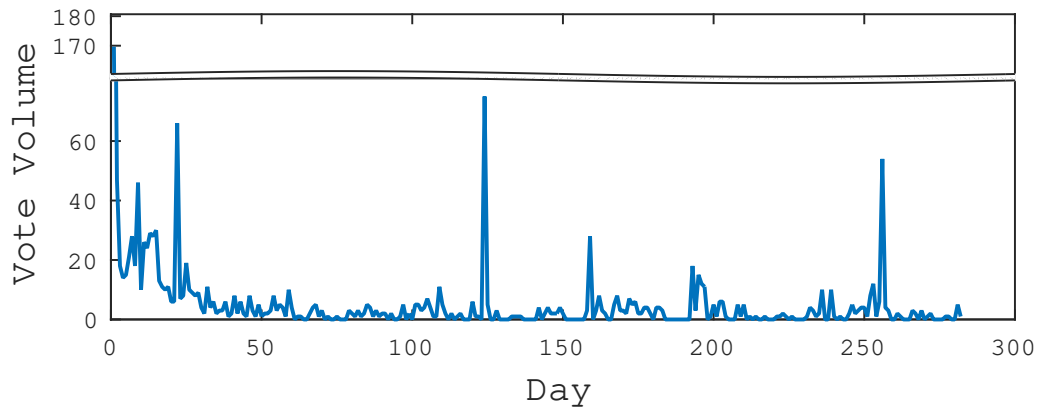


Figure 4.6: Vote volume for each day of study

4.3.2 Primary Study

In the preliminary study, Green and Physical feedbacks consumed the least energy. In addition, Physical feedback received a 100% usefulness vote on our post-study survey, so these two feedbacks were chosen for the Primary Study. Although “Expected Action” was a close third, we chose to limit those tested to maintain a high number of participants in each group. In addition to these two feedback types, we also chose to explore the Drift control strategy, introduced in *Application and Feedback Design*, which would be coupled with the Green feedback on the user applications. These feedback conditions are then assigned to our Primary Study population as described in *Feedback Assignment*

If occupants have perfect knowledge of their ideal temperature, a standard thermostat would help users to maintain comfort. However, as an occupant’s ideal temperature changes with unpredictable factors such as metabolic rate, clothing type, air speed, etc., this is not the case in general. In our pre-study survey, we ask participants for their perceived ideal temperature. As participants vote for their comfort across the study, the building learns their thermal preference. As comfort levels are shown in Table 4.5 to be significantly improved at

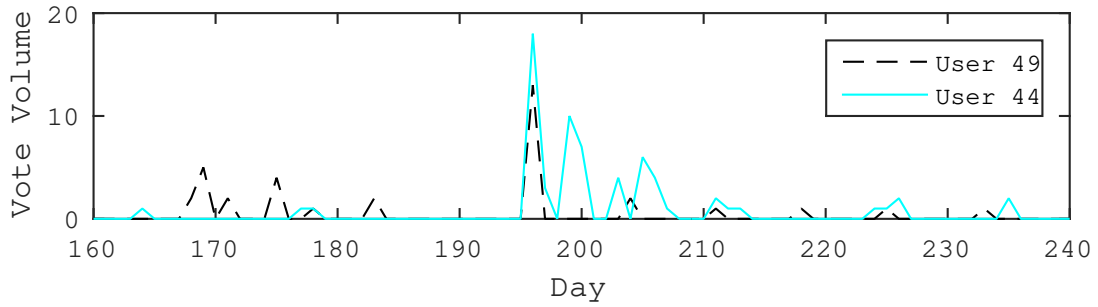


Figure 4.7: Example fault detection usage

the end of the study, we determine the users’ ideal temperature by examining their average room temperature across the last week of the study. The perceived ideal temperatures and the actual ideal temperatures are shown in Figure 4.5. As shown, survey results suggest that occupants prefer temperatures in the range of 61 – 85°F, with a mean preference at 74°F. However, their voting patterns show that the participants’ preferred temperatures fall on the range of 64 – 76°F, with a mean preference at 73°F. The second standard deviations are calculated to be 65°F and 83°F across the survey responses and 67°F and 78°F across the study results. 20% of surveyed users think their ideal temperature is outside of two standard deviations of the actual preferred temperature. If they were in charge of the building, these participants would set the setpoints to an uncomfortable temperature for themselves and others. It is clear from the distributions that users believe they are more comfortable in temperatures further from the mean than they are in practice. Additionally, users believe they would be more comfortable in higher temperatures than their voting patterns indicate (during the summer when outside temperature is higher). This can have an impact on energy consumption, as it is more costly to cool hot outside air to a lower temperature set point.

Figure 4.6 shows how FORCES vote volume changes as the study progresses.

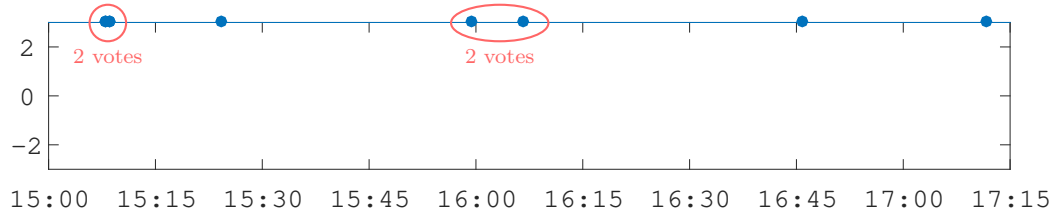
Satisfaction Level	Pre-Survey	Post-Survey
Satisfied	11.9%	66.67%
Somewhat Satisfied	22.0%	26.67%
Neutral	18.6%	6.67%
Somewhat Dissatisfied	28.8%	0.0%
Dissatisfied	18.6%	0.0%

Thermal Comfort	First Vote	Post-Survey
Cold	22.0%	0.0%
Cool	6.0%	0.0%
Slightly Cool	20.0%	12.0%
Neutral	8.0%	60.0%
Slightly Warm	18.0%	12.0%
Warm	6.0%	8.0%
Hot	20.0%	8.0%

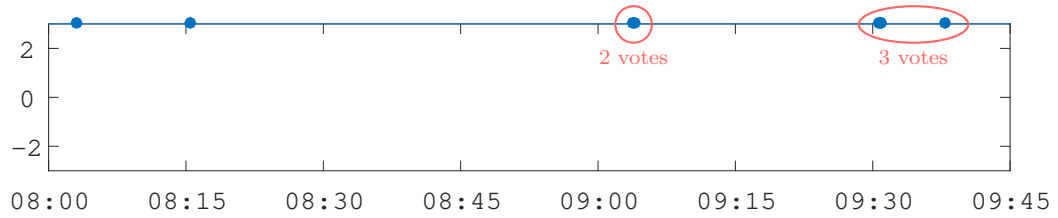
Table 4.5: Satisfaction and comfort

As not all participants could be recruited on the same day, we have shifted the vote timeline so that the day of the study session aligns with “Day 0” in the figure. This also means that votes on the final days represent *only* votes from users that started using the system the earliest. We can see that application usage during the first month is relatively high. As the novelty period wears off and the buildings learn the occupants’ preference, the amount the users vote tends to diminish. Despite the novelty wearing off, we see that FORCES application usage continues into the long-term as users periodically provide corrections to the temperature of their zone. In addition, as usage settles into a background voting pattern, spikes in occupant voting patterns often signify faults in the HVAC system, and can be easily used for fault detection and diagnostics (FDD). One example is shown in Figure 4.7, where spikes in voting patterns for users 44 and 49, who work in separate cubicles in the same office, are correctly time-aligned. Inspecting the data, we found that these peaks correspond to BLD2 HVAC maintenance, where the system was being intermittently disabled.

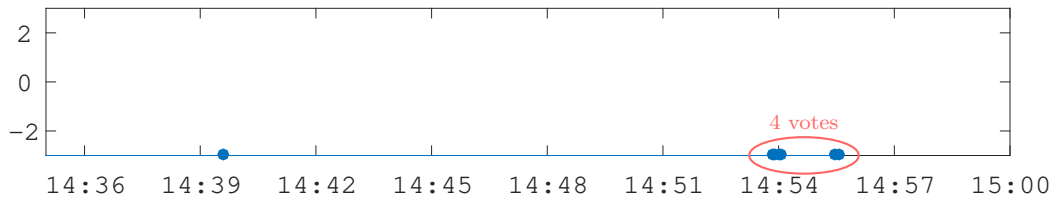
In previous data, we often see patterns where users vote for their comfort, wait a short period of time, and then cast the same vote again. We refer to these duplicate votes as “reiterated votes”, an indication that the user is not yet able to feel the effects of the requested conditioning, and is submitting additional votes to reiterate their request. Three examples of reiterated votes in voting patterns of users with various feedback types can be seen in Figure 4.8. The amount of time required for a room to be conditioned to temperature following a vote depends on the size of the room, amount of ventilation, and other factors. Vote reiteration occurs because it can often take on the order of 15 minutes for an office to reach a desired temperature setpoint, even to perform only a few degrees of temperature change.



(a)



(b)



(c)

Figure 4.8: Reiterated votes

The benefit of physical feedback is the immediate HVAC action in the voter’s space. Although physical feedback does not necessarily help the room reach its desired setpoint significantly faster, the actuation will make clear to the voter that the vote has been received, and the space is being conditioned. If this feedback makes clear to voters that the room is quickly responding to a vote, our data should show fewer reiterated votes from users with physical feedback than with any of the non-physical feedbacks. To determine this, we separate the voting patterns of each individual voter, and find all occurrences of two votes being submitted within some Δt of each other. This Δt is chosen to be twice the system actuation period, or 10 minutes in our experiments. This is because in the worst case, the user votes at the very beginning of a voting period, waits 5 minutes for the vote tally and beginning of the next actuation period, and then waits another entire system actuation period for the room to change to the user’s liking. If a user feels the need to re-vote within this period, we consider it a reiterated vote, caused by the slow response of the system. The results of this analysis can be seen in Table 4.6, where our 23 users with physical feedback are shown to reiterate their votes in 80 clusters, and the remaining 38 users with non-physical feedback cast 377 reiterated votes. Across the 40-week study, users with non-physical feedback cast, on average, $2.9\times$ more reiterated votes than users with physical feedback, supporting the hypothesis that the effects of physical feedback provide immediate feedback that help to quickly satisfy the occupants.

The pre-study survey allowed us to see the initial conditions of the users and their space. At the conclusion of the study, we note what changes the application has made with the comfort and satisfaction of the users.

The thermal comfort at the beginning of the study as shown in Table 4.5 was distributed; 48% of participants were cold to some degree, 44% were hot to

	Total	Avg/User
Physical	80	3.48
Non-Physical	377	9.92

Table 4.6: Reiterated votes for (Non-)Physical feedbacks

some degree, and only the remaining 8% were neutral. As the study progressed and the thermal state changed, the number of neutral (comfortable) participants increased to 60%. Interestingly, no participants reported feeling cold or cool at the end of the study, and the number of those feeling hot dropped from 20% to 8%. The reduction in the extremes, cold, cool, and hot, shows that the thermal status of the room had made a large shift towards an equilibrium around the users' comforts. Additionally, this reduction in extreme votes shows a strong reduction in the amount of over-conditioning occurring in the building, undoubtedly making a strong impact on the energy profile of the building.

Users were asked before and after the study about their overall satisfaction as shown in Table 4.5. Of the 47.4% that claimed to be dissatisfied to some extent at the beginning of the study, none remained dissatisfied at the end. Only 33.9% of their participants considered themselves at least "Somewhat Satisfied" at the beginning, increasing to a much-improved 93.3% by the end. In the pre-study, 33% of participants reported actively avoiding their office due to uncomfortable temperatures, and 96% reported taking individual action to improve their comfort before FORCES. At the end of the FORCES study, none of the participants reported actively avoiding their office, and a reduced 63.3% took individual action to improve their thermal comfort.

In addition to general satisfaction with FORCES, we wish to know how the various feedback types affect user satisfaction. Table 4.7 shows the reported sat-

Satisfaction Level	Green	Physical	Green+Drift
Satisfied	66.67	75.00	62.50
Somewhat Satisfied	0.00	25.00	37.50
Neutral	33.33	0.00	0.00
Somewhat Dissatisfied	0.00	0.00	0.00
Dissatisfied	0.00	0.00	0.00

Table 4.7: Satisfaction of Primary Study Feedbacks

Feedback Type	Baseline Energy (KWH/day)	Feedback Energy (KWH/day)	Percentage Decr.	Projected Cost Decr. (USD)
Green+Drift	95.83	60.27	37.11%	\$93,113.80
Green	54.70	52.31	4.38%	\$10,984.34
Physical	88.65	71.81	18.99%	\$47,663.29

Table 4.8: Energy Cost per Feedback - Primary Study

isfaction after the primary study was complete. Physical feedback resulted in the highest user satisfaction, likely caused by the instant thermal gratification experienced by the user. The Green+Drift feedback resulted in the lowest satisfaction, but all users were still at least “Somewhat Satisfied”, much improved from the pre-study conditions.

As explained in *Feedback Assignment*, each participant in a zone shared the same feedback type for our primary study. Following the procedure described in *Calculating Energy*, the change in energy consumption is computed for each zone, which is then averaged for each feedback type. As shown in Table 4.8, FORCES operation caused a decrease in energy usage for all feedback types, with Green feedback using the most energy at just a 4.4% reduction, and Green+Drift strategy obtaining a 37.1% decrease in energy usage. Although Physical feedback was found to consume more energy than Green feedback in the short-term preliminary

study, analysis of the long-term study data found Physical to result in significant energy savings over Green feedback alone.

To show how the energy savings found in the Primary study would affect a building's energy bill, we project the energy reduction computed for each feedback types onto an example building on our campus, Building 0 (BLD0). BLD0 used 1,091,429 ton-hr for cooling and 124,860.76 therms for heating in 2014. As our campus pays \$0.175(USD)/ ton-hr for cooling and \$0.14(USD)/ therm for heating, \$250,933.24 was spent to condition BLD0 for 1 year. The projected change in cost is reported in the last column of Table 4.8.

4.4 Discussion

The energy model used for energy analysis does not take into consideration the additional wear on the machinery that *may* occur in the system, potentially increasing overall capital costs. Changes in the fan speed and damper position occur over large periods of time and causing a burst of air to be provided to a room may cause a reduction in the life of the system. This additional cost may be considered negligible but further research should be done to confirm this assertion.

In the baseline energy consumption used for comparison, we did *not* include the energy used by occupants using their own personal electrical resistive heaters or fans to improve their personal comfort (a significant number of users based on our pre-study survey data), so the energy savings results reported are conservative.

In our preliminary study, all feedback types showed an increase in energy consumption with respect to the baseline. We believe this was an artifact of the

very few zones that could be analyzed due to the random feedback assignments strategy used in the preliminary study, as it was explained in *Energy Calculation*. Furthermore, the results shown in Fig. 4.4 using all user data would indicate that at least Green feedback should have produced a decrease in energy consumption from baseline. Nevertheless, the results allow us to establish a *relative* comparison among the different feedback types. In the primary study, *every* zone was used in the energy calculation, since each zone was assigned just *one* feedback type. We believe the feedback assignment in the primary study combined with the longer experimental time allowed us to get more meaningful results.

In the primary study, all feedback types decreased energy consumption while improving user satisfaction. Physical feedback saved more energy than Green, but less than Green+Drift. We hypothesize that this may be the result of the human perception of control, as psychological studies have also shown that direct control can lead to greater satisfaction [Pac90]. Users may be willing to vote and/or allow the system to drift to a slight discomfort area, but still be satisfied with the system since they are in control. Although Green+Drift led to significant energy savings, satisfaction was only slightly lower than that of Green and Physical, and higher than baseline.

We believe the results in this work open new potentially exciting interdisciplinary research avenues. By including the human-in-the-loop, we need to further investigate the intersection of human incentives structures (behavioral economists and cognitive scientists) together with control of large cyberphysical systems (engineers) to further understand the inter-relationships between human behavior and system control.

4.5 Summary

Comfort voting applications are becoming more prevalent, but work has not been done to examine how application feedback can affect HVAC energy consumption and occupant satisfaction. In this work, we developed a multi-platform comfort voting application with 5 different methods of feedback. Through a 1-month preliminary study, we investigate how these feedback types affect the satisfaction of the users and the energy consumption of the building. We then use the results to narrow down on the two most promising feedbacks, for evaluation for a long-term study. Across a 40-week study covering 3 buildings and 61 participants, we find that using feedback systems can improve satisfaction from 33.9% to 93.3%, and reduce energy consumption by 18.99%. Furthermore, with the inclusion of a drifting control strategy, a 37% energy reduction can be realized without significantly reducing occupant comfort.

CHAPTER 5

Building CoolUs: Control using Optimization, Occupancy, and Local User Sensing

The previous two chapters were focused on comfort and occupancy as two independent concepts, but using a model predictive control (MPC) scheme these two techniques can be combined into a larger problem. By using both occupant and comfort data, we can use the data as in an MPC problem and develop an optimization problem. With an optimization based solution based on the entire HVAC system and occupancy patterns we can minimize monetary cost through out the building while constrained to occupant comfort. In this chapter we present a model predictive control (MPC) framework for smart building control. The framework has several components, including (a) occupancy sensing in real-time, (b) occupancy prediction models based on historical occupancy data, (c) thermodynamic building models, participatory sensing model and (e) a model predictive control optimization that minimizes monetary costs in energy use while maintaining quality comfort bounds for the building's users. The occupancy sensing information is based on the methods developed in Chapter 3 and the participatory sensing method as explained in Chapter 4

The main contributions offered in this chapter are:

- A Model Predictive Control (MPC) framework that considers both occupancy prediction and participatory sensing

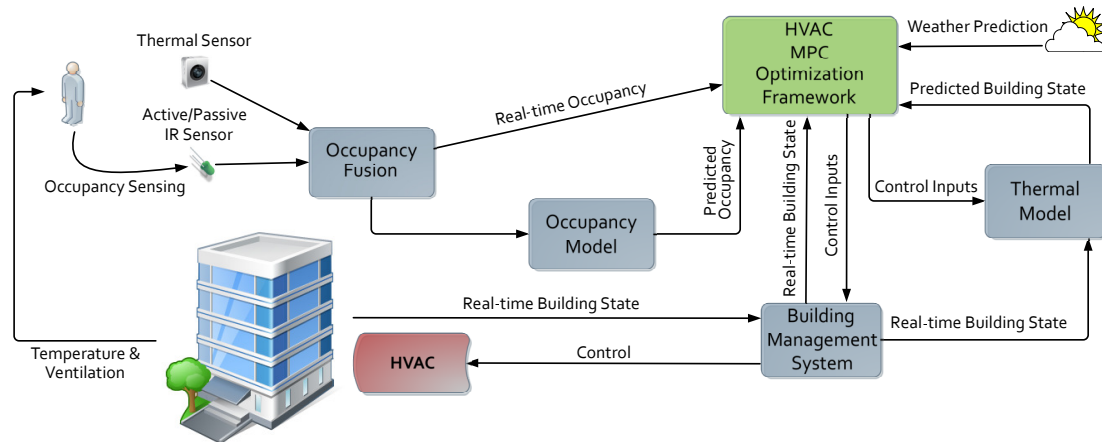


Figure 5.1: CoolUs Architecture Overview

- An experimental evaluation on a real production building with detail analysis of cost reductions

5.1 System Overview

Our system periodically collects data from sensors throughout the building and outside environment in order to find an optimal control strategy that minimizes monetary cost while satisfying occupant’s comfort. An important component of our architecture is the use of thermal and building’s occupancy models that allow us to predict the behavior of the building and users and use these predictions for optimal control.

Figure 5.1 shows an overview of our system architecture. Sensor data from the building is collected by the Building Management System (BMS) periodically throughout the day. The data collected are standard sensors in buildings such as thermostats, flow and temperature sensors throughout the HVAC system. These sensor values are collected directly from the BMS’s SOAP based API and using the BACnet [BAC99] protocol and stored on a sMAP archiver [DJT10b]. Along

with the standard sensors in the building, we use occupancy detection sensors and prediction models as outlined in Chapter 3. Additionally, we use local weather forecast to gather the outdoor temperature, which affects the thermodynamics of the zone. To ensure that we are keeping occupants comfortable we include the participatory voting system that was introduced in Chapter 4.

With a combination of the occupancy prediction, initial conditions, and weather prediction we can solve the MPC formulation described in Section 5.2.4 below. Once the optimizer solves the problem, a set of control inputs is obtained, such as the mass flow and discharge temperature of *each* zone. These control set points are then transmitted to the BMS, which then uses its own built-in PID loops to achieve the set points passed to the building's equipment.

5.2 Model Predictive Control

5.2.1 HVAC System Description

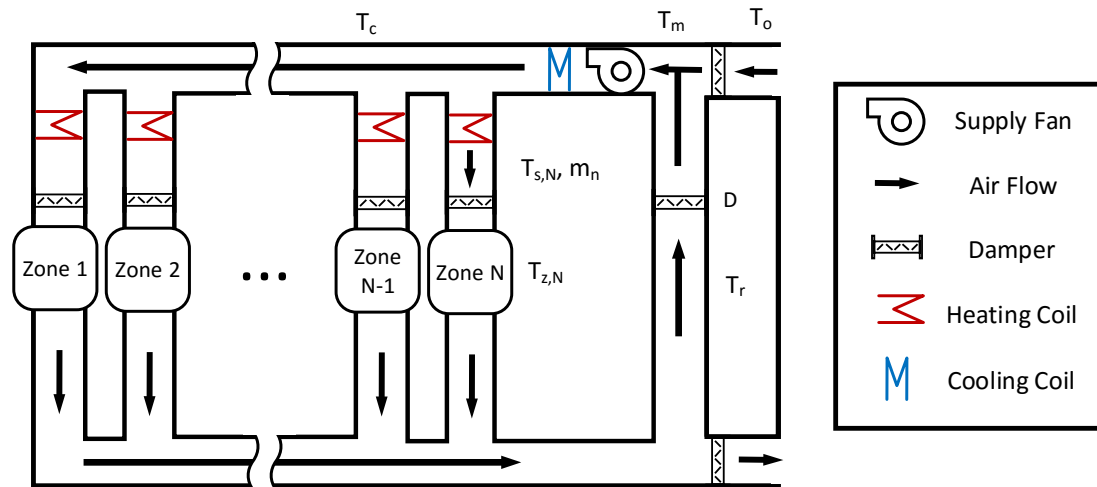


Figure 5.2: HVAC Single Duct VAV Terminal Reheat Layout

The HVAC system we modeled is a single duct central cooling HVAC with

terminal reheat as shown in Figure 5.2. The process begins at the supply fan in the air handler unit (AHU), which supplies air for all of the zones. The supply fan's air first goes through a cooling coil, which cools the air to the minimum required temperature needed for all of the zones. Before air enters a zone, the air passes through a variable air volume (VAV) unit that regulates the amount of air that flows into a zone. Terminal reheat occurs when the heating coil increases the temperature before discharging air into a zone. A discharge setpoint temperature is selected for each zone and the VAV ensures that the air is heated to this temperature for each zone. The air supplied to the zone is mixed with the current zone air, and some of the air is exhausted out of the zone to maintain a constant static pressure. The return air from each zone is mixed in the return duct, and then portions of it may enter the economizer.

The economizer unit contains several dampers to regulate the amount of air that is recirculated in the system and the amount of fresh air that is brought into the system from the outside. At certain times of the year, it is viable to use all outside air if the outside air temperature is close to the desired set point. It is also possible that the outside air temperature is significantly higher or lower than the desired temperature at which point most of the air is recirculated inside the building to save energy. Although it may be desirable to recirculate all possible air in the system to reduce cooling and heating costs, there is a necessary minimum amount of outside air required for a zone due to carbon dioxide and other gases generated by furniture, appliances and occupants of the zone. Once the air is mixed, it returns back to the air handling unit and the cycle begins again.

Type	Symbol	Description
Constants	M_n	Capacity of a zone
	c_p	Heat capacity of air
	$R_{n,i}$	Resistance between zone i and zone n
	$R_{n,o}$	Resistance between zone n and the outside
	η_c, η_h	Cooling and heating efficiency
	r_g	Monetary cost of gas per kwh
	r_e	Monetary cost of electricity per kwh
	τ, N	Number of time steps and zones
	$T_{c,min}(t)$	Min cooling temperature
	$T_{s,max}(t)$	Max heating temperature
	R_o	Required ventilation per occupant
	R_{a_n}	Required ventilation for an area
	A_n	Area of a zone
Variable	$T_{z_n}(t)$	Zone Temperature
	$T_{s_n}(t)$	Supply air temperature to a zone
	$T_c(t)$	Cooled air temperature supplied by the AHU
	$T_m(t)$	Mixed Air Temperature
	$D(t)$	Damper position for outside air
	$T_r(t)$	Return temperature
	$m_{z_n}(t)$	Mass flow into a zone
	$m_s(t)$	Total mass flow of all zones
	$\rho_{u,n}(t), \rho_{l,n}(t)$	Upper and lower comfort penalty parameters
	ϕ_p	Penalty coefficient
	t, n	Time, Zone
Given or Pre-calculated	$T_o(t)$	Outside air temperature
	Q	Thermal load
	$O_n(t)$	Occupancy
	$V_{min,n}(t)$	Minimum ventilation required for a zone
	$T_{z_n}^+(t), T_{z_n}^-(t)$	Upper and lower comfort bounds

Table 5.1: MPC variables and descriptions

5.2.2 Objective Function

Our optimization problem, at a high level, can be seen as the minimization of HVAC operation’s monetary cost subject to occupant comfort, regulatory constraints, and hardware constraints. For MPC, we need to insert our initial state of the building, and model the changes in thermodynamics of the room over a set period to ensure we conform to the constraints of the system. Time is a crucial part of our formulation due to changes in occupancy, which require pre-conditioning a zone in time for an occupant who we predict is about to enter a zone. Overall we attempt to find the best control sequences while minimizing monetary cost due to energy usage. In our formulation of the system, we aim to keep the system convex by making our objective function quadratic with respect to the optimization variables and only use linear constraints. A convex problem is computationally easier to solve and has no local optima. Our end result, however, is a nonconvex quadratic problem with linear constraints.

In the next sections, several variables and constants will be introduced, which are listed in Table 5.1. In this table, variables labeled as **Given** may vary across time, but can be pre-calculated or fetched before the optimization problem starts. An example of a **Given** function is the outside temperature, $T_o(t)$, which can be acquired via weather forecasting sources. See Section 5.2.4 for further explanation.

The monetary cost of the HVAC system used in our objective function is the sum of the power used to cool air, P_c , the power used to heat the air, P_h , and the power used to move the air via the supply fan, P_f . These different power usages are converted to a common unit such as monetary value since each component may draw its power from different energy sources such as gas or electricity, which costs typically vary. We use the conversion constants r_e and r_g to convert power

obtained from electricity and power obtained from gas into a monetary amount, respectively.

First, the cost of cooling is the change in temperature from the mixed air (T_m) to the supplied air (T_c). The efficiency of decreasing the temperature is η_c shown in Equation 5.2

$$m_s(t) = \sum_{n=1}^N m_{z_n}(t) \quad (5.1)$$

$$P_c = \frac{c_p}{\eta_c} \sum_{t=1}^{\tau} m_s(t)(T_m(t) - T_c(t)) \quad (5.2)$$

The mixed air temperature T_m is a mixture of the return temperature T_r and the outside temperature (T_o). The return temperature is based on the contribution of the temperature that comes from each zone. The return air is written as:

$$T_r(t) = \frac{\sum_{n=1}^N m_{z_n}(t)T_{z_n}(t)}{m_s(t)} \quad (5.3)$$

Once the air returns and arrives at the economizer section of the AHU, part of it is exhausted and part is recirculated. The amount recirculated is based on the damper position, $D(t)$, which has a value between 0 and 1. $D(t) = 1$ is when the recirculating damper is fully opened and $D(t) = 0$ when the system is using all outside air.

$$T_m(t) = D(t)T_r(t) + (1 - D(t))T_o(t) \quad (5.4)$$

If we substitute values back to the original P_c , the division of m_s is removed from T_r , but we are still left with a cubic polynomial of optimization variables

in the objective function with $D(t) \sum_{n=1}^N m_{z_n}(t) T_{z_n}(t)$ which increases the complexity of the problem. To reduce the order of this polynomial, we linearize $f(T_{z_n}(t), m_{z_n}(t)) = T_{z_n}(t) m_{z_n}(t)$. Equation 5.3 can therefore be re-written as:

$$T_r(t) = \frac{\sum_{n=1}^N -\bar{T}_{z_n} \bar{m}_{z_n} + \bar{m}_{z_n} T_{z_n}(t) + m_{z_n}(t) \bar{T}_{z_n}}{m_s(t)} \quad (5.5)$$

By linearizing this term, we keep our objective function quadratic. We minimize the error of this model simplification by choosing sensible values of \bar{m}_{z_n} and \bar{T}_{z_n} , as explained in Section 5.4.1.

Similarly the cost of heating is the change in temperature from the cool air supplied by the AHU to the supplied air temperature. The supplied temperature is different per VAV box so the mass flow into each zone is separated to find the cost per zone. The efficiency of heating this air, η_h , is then applied to the sum of all the zones.

$$P_h = \frac{c_p}{\eta_h} \sum_{i=1}^N \sum_{t=1}^{\tau} m_{z_i}(t) (T_{s_i}(t) - T_c(t)) \quad (5.6)$$

The cost of the fan is shown to be a squared increase with respect to mass flow in [KMB13] and is written in Equation 5.7 with an efficiency factor of η_f .

$$P_f = \sum_{t=1}^{\tau} \eta_f m_s(t)^2 \quad (5.7)$$

Due to changes in occupancy, there may be periods of time when a single time step is not enough time for our zone temperature to fall within the feasible range. This can occur due to an incorrect occupancy prediction which may leave the room in a very warm/cold state when an occupant enters the room. To relax our comfort constraint we include a penalty function ρ_{u_n} and ρ_{u_n} . These

penalty function are included into our cost function and also included in our comfort constraint as discussed in Section 5.2.3.1. This penalty is added to our formulation as shown in Equation 5.8. We multiply this constraint by a coefficient, ϕ_p , to denote how strict we want the bounds to be.

$$C_s = \phi_p \sum_{n=1}^N \sum_{t=1}^{\tau} \rho_{u_n}(t) + \rho_{l_n}(t) \quad (5.8)$$

Simply summing the power terms does not capture the difference of pricing between different forms of energy, specifically the differences between using gas and electricity. In our location, gas is significantly cheaper than electricity. We therefore have a conversion to a monetary cost with conversion factors r_e and r_g for electricity and gas, respectively. The heating element is based on gas and both the fan and cooling are based on electricity. Along with the monetary cost, we add our comfort penalty, C_s . The final objective function is therefore:

$$\min_{\substack{m_n(t), T_{z_n}(t), T_{s_n}(t), T_c(t), D(t) \\ t \in 1 \dots \tau, n \in 1 \dots N}} r_e(P_c + P_f) + r_g(P_h) + C_s \quad (5.9)$$

subject to: Constraints 1-12

At this point, we have a quadratic nonconvex objective function. Our problem is non-convex due to the formulation of P_c and P_h . Both have a term that are similar in structure $(x-y)z$ which makes the problem nonconvex even though our constraints are $x-y \geq 0$. Specifically for equation 5.2 we have $m_s(t)(T_m(t) - T_c(t))$. In the following section, we will introduce linear constraints to our problem.

5.2.3 Constraints

5.2.3.1 Thermal Model Constraint

The following thermal model proposed is heavily influenced by [KMB13] with modifications to make the constraint linear. In order to model the thermodynamics of a zone, we use a resistor-capacitor model where each zone is considered a capacitor and interactions between zones are resistors. Some of the variables in our formulation are overlaid on Figure 5.2 to help show the physical relationship these variables have with the AHU loop. Each zone n has a capacity of M_n and, between zone n and zone j , a resistance $R_{n,j}$. The resistance between a zone and the outside temperature is considered a special case noted as $R_{n,o}$.

To model the change in a zone n 's current temperature, T_{z_n} , we break apart the problem into a few core components include the displacement of air and the heat transfer of zones. Each zone is supplied air, m_{z_n} , at a supply temperature of T_{s_n} . If m_{z_n} air is put into a zone, an equal amount of air within the zone must exit the system to maintain a constant static pressure. This transfer of air can be represented as $(T_s - T_z)c_p m_z$ where c_p is the air capacity. The heat transfer that occurs between a zone and an outside wall is $(T_o - T_{z_n})R_{n,o}$ where T_o is the outside temperature. In our model, the heat transfer that occurs between adjacent zones is $\sum_{i=0}^N (T_{z_i} - T_{z_n})/R_{n,i}$. Along with this, there is a thermal load Q_n in the zone. This load can be attributed to occupancy, appliances, machinery, and objects that increase/decrease the temperature. The general thermal model, based on [KMB13], for this system for a single zone n is:

$$M_n \frac{d}{dt} T_{z_n} = (T_{s_n} - T_{z_n})c_p m_{z_n} + Q_n + (T_o - T_{z_n})/R_{n,o} + \sum_{i=0, i \neq n}^N (T_{z_i} - T_{z_n})/R_{n,i} \quad (5.10)$$

We only considered the outside temperature due to the larger temperature differential between a conditioned zone and outside weather, which causes a larger change in temperature in a zone which leads the term $(T_{z_i} - T_{z_n})/R_{n,i}$ to approach 0. Our equation can therefore be simplified to:

$$M_n \frac{d}{dt} T_{z_n} = (T_{s_n} - T_{z_n}) c_p m_{z_n} + Q_n + (T_o - T_{z_n}) / R_{n,o} \quad (5.11)$$

Due to m_{z_n} , T_{s_n} , and T_{z_n} being optimization variables, the first term in Equation 5.11 causes the constraint to be non-linear. These constraints can be made linear using Taylor's theorem as in Equation 5.12 where \bar{T}_{s_n} , \bar{T}_{z_n} , and \bar{m}_{z_n} are initial conditions for T_{s_n} , T_{z_n} and m_{z_n} , respectively, for all values of t . Selection of good \bar{T}_{s_n} , \bar{T}_{z_n} , and \bar{m}_{z_n} is discussed in Section 5.4.1.

$$\begin{aligned} M_n \frac{d}{dt} T_{z_n} = & c_p (-\bar{T}_{s_n} \bar{m}_{z_n} + \bar{m}_{z_n} T_{s_n} + m_{z_n} \bar{T}_{s_n} + \bar{T}_{z_n} \bar{m}_{z_n} - \bar{m}_{z_n} T_{z_n} - m_{z_n} \bar{T}_{z_n}) \\ & + Q_n + (T_o - T_{z_n}) / R_{n,o} \end{aligned} \quad (5.12)$$

Equation 5.12 can be discretized with respect to time as follows:

$$\begin{aligned} T_{z_n}(t+1) = & T_{z_n}(t) + \frac{\Delta t}{M_n} (c_p (-\bar{T}_{s_n} \bar{m}_{z_n} + \bar{m}_{z_n} T_{s_n}(t) + m_{z_n}(t) \bar{T}_{s_n} + \bar{T}_{z_n} \bar{m}_{z_n} \\ & - \bar{m}_{z_n} T_{z_n}(t) - m_{z_n}(t) \bar{T}_{z_n}) + Q_n + (T_o(t) - T_{z_n}(t)) / R_{n,o}) \end{aligned} \quad (5.13)$$

This Equation will be considered as our first constraint of the system.

Constraint 1: Equation 5.13 – Thermal Model

The remaining constraints are based on the physical properties of the system, building regulations that the HVAC system must follow, and comfort constraints.

Constraint 2: $T_c(t) \leq T_m(t)$ – The cooling coil can only cool the air received from the economizer.

Constraint 3: $T_c(t) \geq T_{c,max}$ – Cooling capacity of cooling coil.

Constraint 4: $T_s(t) \geq T_c(t)$ – The heating coil can only increase the temperature of the air supplied by the AHU.

Constraint 5: $T_s(t) \leq T_{s,max}$ – Max capacity of heating coil.

Constraint 6: $m_{z_n}(t) \geq V_{min,n}(t)$ – The zone’s minimum ventilation required for occupants. (See Section 5.2.4)

Constraint 7: $m_{z_n}(t) \leq m_{max_n}$ – The VAV’s maximum ventilation capacity.

Constraint 8: $0 \leq D(t) \leq 1$ – Physical damper constraint.

Constraint 9: $D(t) \geq D_{min}$ – Minimum outside air requirement.

Constraint 10: $\sum_{n=1}^N m_{z_n}(t) \leq AHU_{max}$ – The fan’s maximum ventilation capacity.

Constraint 11: $\rho_{u_n}(t) + T_{z_n}^+ \geq T_{z_n}(t) \geq T_{z_n}^- - \rho_{u_n}(t)$ – Comfort bound

Constraint 12: $\rho_{u_n}(t) \geq 0, \rho_{t_n}(t) \geq 0$ – Penalty functions can only increase cost

5.2.4 Given/Precalculated Constants

Using the HVAC efficiently is an important part of energy savings, but savings can be easily achieved by simply turning a building’s conditioning off. In order to properly evaluate the system, the zone temperature should fall within the temperature bounds proposed in ASHRAE Standard 55 [ASH04]. In ASHRAE 55, comfort is defined as a Predicted Mean Vote (PMV) which is a scale from -3 to 3, where -3 indicates occupants are cold and +3 indicates occupants are

Parameter	Values
Mean Radiant Temperature	$75^{\circ}F$
Humidity	30 %
Metabolic Rate	$1.1 W/m^2$
Clothing	$0.8 Km^2/W$
Air Speed	$0.1 m/s$

Table 5.2: Parameters used for PMV

hot. An occupant is considered to feel neither too hot or too cold when PMV is between $-0.5 < PMV < 0.5$. In the calculation of an ideal temperature we used the parameters found in Table 5.2. These values contain the humidity of our region, the metabolism of office workers, and a clothing coefficient that is approximately that of spring time, when our experiments took place.

As discussed in Chapter 4, PMV does not properly reflect how a user actually feels, so users are encouraged to provide feedback to the system using our voting application. We use the leaderboard strategy as discussed in Chapter 4 since it was found to encourage users to vote, but not necessarily vote in an energy conscious mindset. When a user votes on their preference, we assume that their preference does not change during the time horizon of the prediction. Therefore each zone has a upper comfort bound, $T_{c_n}^+$, and a lower comfort bound , $T_{c_n}^-$ based on the occupants last vote.

Meeting AMV standards are only required when the zone is occupied. Therefore, we have two constraints based on the occupancy prediction, $O_n(t)$, where n is the zone. The unoccupied bounds are the constants, $T_{zu}^+(t)$, $T_{zu}^-(t)$. This is written in Equation 5.14 and 5.15.

$$T_{z_n}^+ = \begin{cases} T_{c_n}^+ & \text{if } O_n(t) > 0 \\ T_{z_u}^+ & \text{if } O_n(t) = 0 \end{cases} \quad (5.14)$$

$$T_{z_n}^- = \begin{cases} T_{c_n}^- & \text{if } O_n(t) > 0 \\ T_{z_u}^- & \text{if } O_n(t) = 0 \end{cases} \quad (5.15)$$

Alongside comfort requirements, there are also ventilation requirements to reduce gases that build up in a room. To fall within ASHRAE Standard 62.1 [ASH07] the zone must be properly ventilated. By knowing the number of occupants we can find the exact minimum required cubic feet per minute (CFM) for each zone. The amount of CFM required is shown in Equation 5.16, where R_p is the minimum CFM per person, R_a is CFM per ft^2 , P_z is the number of occupants and A_z is the zone's area. For an office the $R_p = 5CFM/person$ and $R_p = 0.6cfm/ft^2$. The square footage of each zone is [275, 725, 725, 400, 300, 700, 400, 400], respectively. A conversion from CFM, which is a unit of volume, to kgps is approximated by using air pressure at sea level at $70^\circ F$.

$$V_{min,n}(t) = R_p O_n(t) + R_a A_n \quad (5.16)$$

The final precalculated value in our system that is retrieved is weather forecasting used for T_o . Forecasts are retrieved from the OpenWeatherMap API [Ope].

5.3 Implementation Details

In order to solve the model described in the previous sections we used the programming language Julia [BEK17] using the optimization library JuMP [DHL17]. JuMP allows for different solvers to be interchanged with relative ease. To solve our quadratic objective problem with linear constraints, we use an Interior Point

Optimizer (Ipopt) [WB06], which is standard solver and capable of solving non-linear optimization problems. To solve a problem which models a building of size $N = 9$ and time steps $\tau = 6$ takes on average 0.2 seconds. As building actuation is done on the scale of minutes, the time it takes to solve for a single optimization problem is well within what is considered acceptable.

Every 5 minutes, we solve the optimization problem and gather the solved variables to be actuated. Within the buildings management system (BMS), we can set our desired points for each variable through out the building and allow the normal building control routines to take control. An example of how we control would be setting the point for the mass flow. For each zone, our solution contains the flow for each zone which can then be set as required mass flow can be set via the BMS. We only adjust the setpoint, and the building is capable of changing the supply fan's speed and damper position for each zone to meet the needs of all the zones. Another possible way of control, is to include the damper position and fan speed within our model, however this adds additional complications on the number of setpoints required to control the building and requires to add additional complexity to the model. It is also possible that these lower level setpoints are not available to be controlled by the BMS interface. Because of these reasons we chose to use the high level set point method. The points we set every actuation period include all zones' mass flow (m_{z_n}), zones' supply temperature (T_{s_i}), and cooling temperature (T_c). In order to avoid issues with static pressure, we do not modify the outside air damper position and allow the BMS to use it's standard economizer control routines.

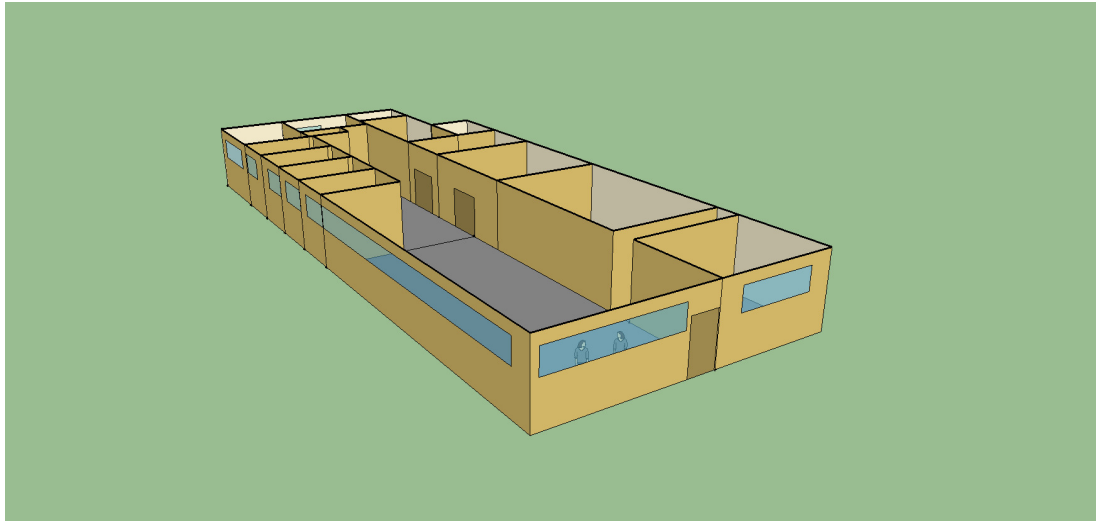


Figure 5.3: 3-D Model of building

5.4 Evaluation

Evaluation of this model was conducted at the University of California, Merced campus during the summer. Experiments were run during the summer where the temperature were at lows of $61^{\circ}F$ to highs $96^{\circ}F$. We chose a single floor office space with a square footage of $5750ft^2$, as depicted in Figure 5.3. This building contains 40 workers that occupy the building between 6:00am and 2:00am.

5.4.1 Model Evaluation

In Section 5.2.3.1, a non-linear thermal model and linear thermal model was established. In this section we confirm that both models are accurate to what occurs in an actual building.

The unknowns in a building would be the capacitance, M_n , the resistance, and the thermal load, Q , of each zone. Based on the thermal model alone, each zone's constants are independent to each other, which makes solving these values

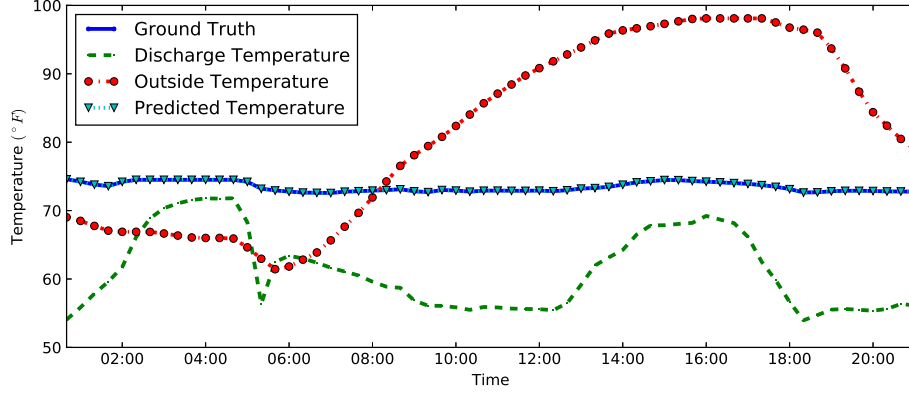


Figure 5.4: Non-Linear Thermal Model

possible by running a regression using historical data. Which data is chosen is crucial since during normal operations of the HVAC system the change in temperature will be near zero between each time step. It is therefore advantageous to use the warm-up period at the start of the day when the system is at its coolest for our training data. At the start of the day, the system is trying to reach the desired set point as quickly as possible. The above parameters could also be more accurately trained by perturbing the system causing large changes in temperature.

When making the original thermal model into a linear problem, we had to generalize the relationship using a first order Taylor series. For this linear function to be accurate, we must choose initial points near the range we are evaluating the function. For both zone temperature and supply temperature, this is relatively easy. The change in a zone temperature will be small through out each time step so \bar{T}_{z_n} was set to be equal to $T_{z_n}(0)$. For supply temperature, we can make the assumption that during the summer this temperature will be in cooling mode and near $55^\circ F$ and for heating $80^\circ F$. The mass flow \bar{m}_{z_n} is probably the most difficult parameter to choose since it can range from $V_{min,n}(t)$ to $m_{max,n}$. Any

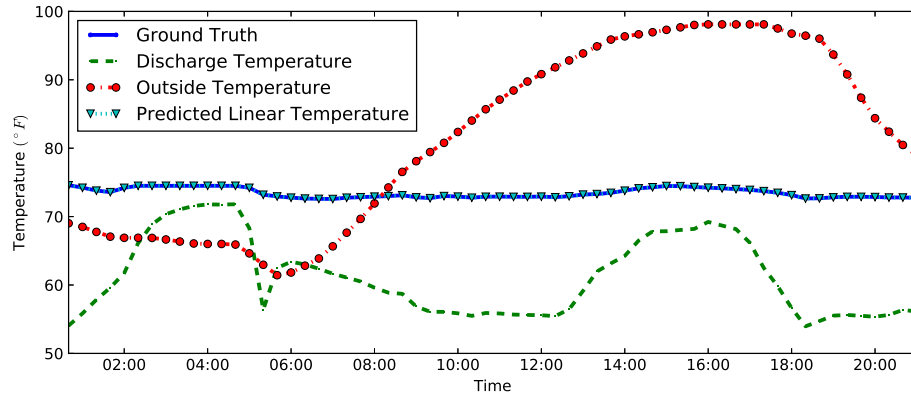


Figure 5.5: Linear Thermal Model

value between these two are valid, but for our test we picked a value at $V_{min,n}(t)$. As stated in Equation 5.16, the minimum ventilation is calculated for each zone dependent on occupancy and size of zone at each timestep. We use the above values when evaluating the linear model.

To evaluate the model’s accuracy, we step through historical data of the building during normal operation and pass known variables such as the supplied temperature, current temperature, and mass flow into our thermal model to find $t + 1$. With historical data we can compare the predicted temperature to the zone’s temperature and determine the accuracy of the model by calculating the Root Mean Square Error (RMSE). The Figure 5.4 shows the non-linear thermal model which follows the ground truth closely. Figure 5.5 is the linear model on the same day as Figure 5.4. The ground truth is the expected temperature in Δt seconds. RMSE for the non-linear and linear thermal models are 0.0120 and 0.0146, respectively. This RMSE means that 95% of data falls within $+/- 2$ times or less than $+/- 0.03^\circ F$ which in our application is more than acceptable.

The previous figures were based solely on historical data during normal modes

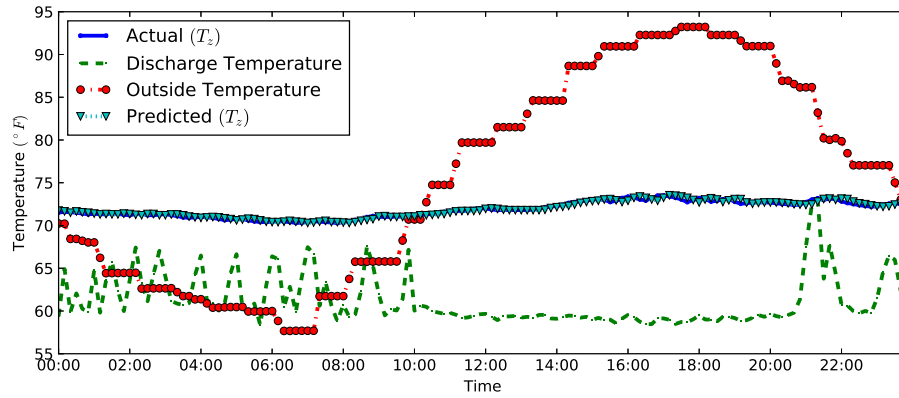


Figure 5.6: Zone 2: Predicted vs. Measured Zone Temperature Results

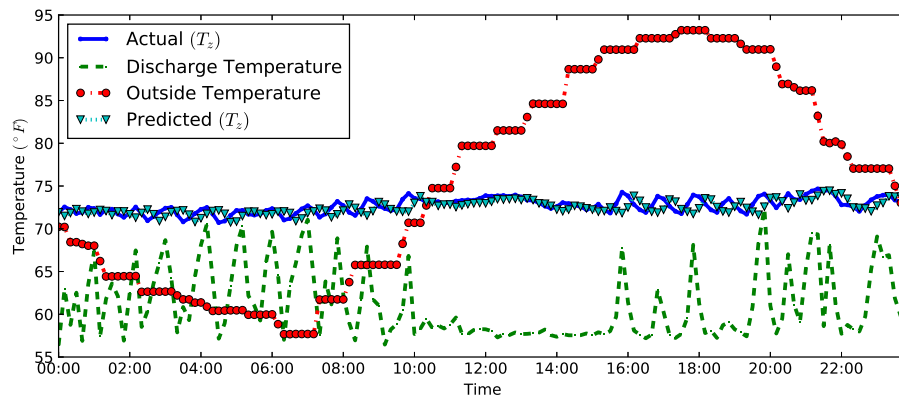


Figure 5.7: Zone 1: Predicted vs. Measured Zone Temperature Results

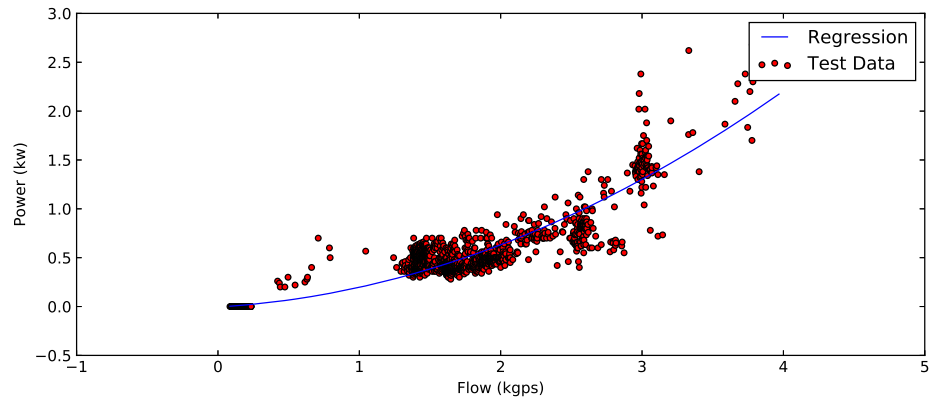


Figure 5.8: Fan quadratic regression

of operation within the building and provides a fairly low error. Once we begin to use the CoolUs system, the expected temperature and actual temperature varied more when compared to historical data. Figure 5.6 and Figure 5.7 show the expected predicted zone temperature and the actual zone temperature using CoolUS. In Figure 5.6, you can see a RMSE of 0.2090, which is greater than the error from historical data, but still makes most predictions ± 0.4180 within the actual temperature. Figure 5.7 does not do as well of a job with the system constantly either undershooting or overshooting the predicted temperature. After inspecting the room, the room is found to have several large appliances including a soda vending machine and a residential refrigerator. Both of these devices have condensers, which transfer heat to the environment and activate periodically. Due to this, the assumption that load is constant is not necessarily true when large thermal load objects are in inside a room. This zone had a RMSE of 0.8651.

Based on the flow values and a power metered fan, we are able to check if the fan power usage is indeed the one presented in Equation 5.7. Figure 5.8 shows a second degree polynomial fit and the testing set of data. The RMSE for this fit

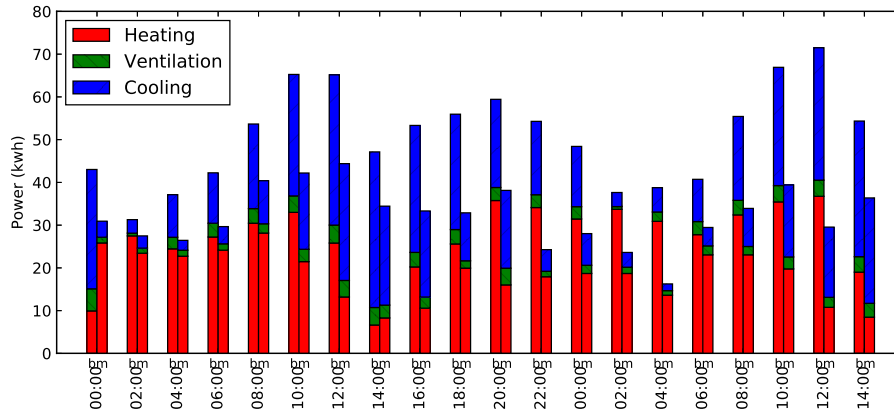


Figure 5.9: Energy comparison between baseline (b) and CoolUs(m)

is 0.9051 which is a reasonable for our application. This is likely due to the high noise during 1.5 kgps to 3 kgps.

5.4.2 Energy Usage

With a well fit model, this section focuses on the energy analysis of a deployment on a real building. We compare energy usages over five days with similar highest temperature and average temperature within the same month in the same building. The first strategy in described as the baseline strategy which is a static schedule control strategy commonly used in buildings. This strategy keeps each zone’s temperature within set bounds assuming full occupancy during the schedule occupied time The system noted as “m” for MPC is using the CoolUs system as described in the previous sections. Figure 5.9 zooms into two days out of the five to show how the energy expenditure fluctuates through out the day.

Figure 5.9 is the breakdown of how energy was used through out the experiment compared to the baseline. In this figure we can see that CoolUs consistently outperforms the baseline in heating, cooling, and ventilation through the entire

System	Heating (kwh)	Ventilation (kwh)	Cooling (kwh)	Total (kwh)
Baseline	1372.88	144.17	1229.81	2746.86
CoolUs	906.43 (-34.0%)	111.14 (-22.9%)	559.17 (-54.5%)	1576.7456 (-42.6%)

Table 5.3: Energy Usage

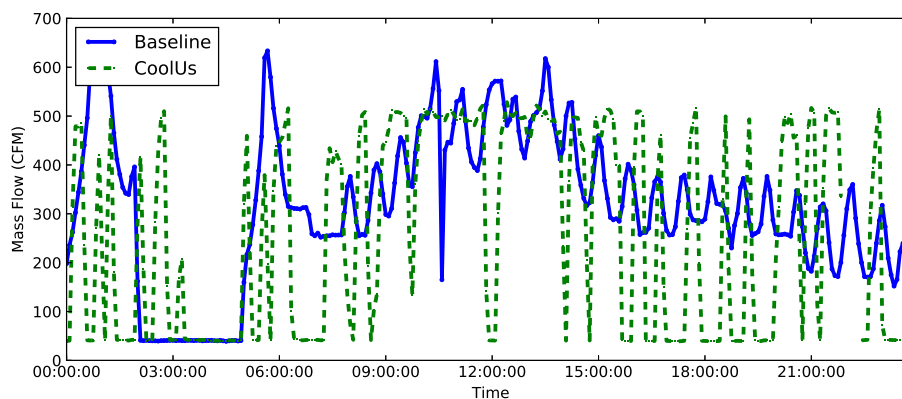


Figure 5.10: Baseline and CoolUs Mass Flow Patterns

day. The energy saved is due to reduction of mass flow blown into the room and a reduction in the temperature of cooling. During our experiments, the baseline used 1170 kg of air while the CoolUs system used 820 kg which is a 30.0% reduction in ventilation. On average using the baseline the cool air temperature is $52^{\circ}F$ and CoolUs uses $62^{\circ}F$ cooled air. Table 5.3 shows the energy usage during the evaluation period of the baseline and CoolUs system over a work week period. The total reduction of the energy usage is 42.6% with the majority of the savings coming from the decreasing in cooling at 54.5%.

A reduction in air flow reduces the amount of air that needs to be cooled. In Figure 5.10, we can see a fluctuation in the amount of ventilation for a single zone, the break room. Peaks occur when the HVAC is attempting to condition a room, while the valleys are period when only minimum ventilation based on occupancy

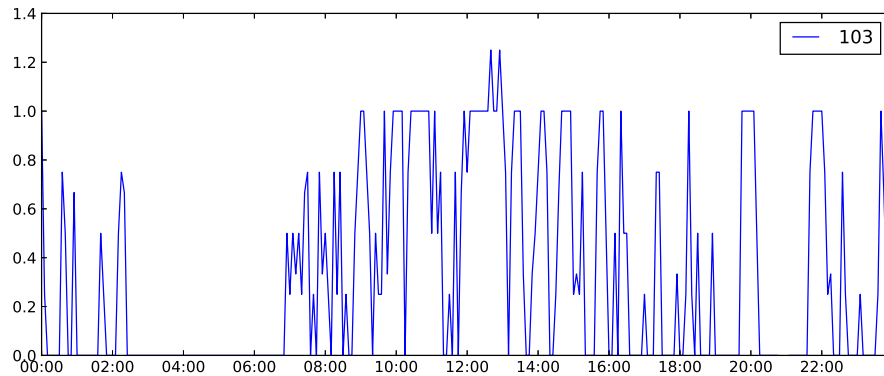


Figure 5.11: CoolUs Occupancy

is used. Figure 5.11 shows the occupancy for the break room and by comparing these two figures side by side, we can see some overlap during extended periods of occupancy such as 2:00am until 6:00am, but the room is being ventilated during unoccupied periods such as 6:00am to 7:00am. This is due our system using a predicted occupancy to precondition a zone. The baseline does preconditioning also, but based on a rule which begins to precondition an hour before occupants are schedule to enter a zone.

5.5 Summary

In this chapter we show that an accurate model can be developed for a building's thermodynamics. This model has a root mean square error of 0.0146 and 0.2090 when compared to historical data and when actually used in a zone with a constant load, respectively. A five day study controlling a building showed that our system can decrease HVAC energy usage by 42.6% using a combination of occupancy detection, comfort voting, and a model predictive control framework.

CHAPTER 6

Conclusion

In this thesis, we covered several topics in the smart building domain. We first started by focusing on energy savings that can be achieved by detecting occupancy using a novel sensing technique. Using this new sensor and our regression pipeline we obtain a root mean square error of 0.35 when evaluated in 10 building zones over three weeks. After discussing occupancy sensing, we focused on thermal comfort of participants and how to influence how occupants participate in thermal voting systems. Several feedback types were evaluated to see which feedback mechanisms can reduce energy cost while maintaining thermal comfort. Using the occupancy sensing and participatory sensing, we combined these two techniques to create a model based approach to reduce monetary expenditure in buildings. The occupancy detection technique was used to ensure the zones within a building were only conditioned when occupied and pre-condition before occupants enter a space. The comfort data we used was capable of adapting the thermal constraints of a zone depending on how they voted on their thermal comfort. By combining these two methods, we were able to reduce energy expenditure by 42.6% during our week long evaluation using our model predictive approach.

Based on this work, there are many avenues for future work to be taken. The optimization problem developed in the end can be further expanded to take additional elements into consideration such as non-constant thermal load. There are also stochastic model prediction methods that can be explored to take the

probabilistic nature of changes in occupancy and comfort. This stochastic model would also require work towards a probability model of human comfort.

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